

FATIGUE CRACK GROWTH PREDICTION AND ANALYSIS

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Turgut AKYÜREK

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FATIGUE CRACK GROWTH PREDICTION AND ANALYSIS

APPROVED BY

Dog.Dr. Öktem VARDAR (Thesis supervisor)

Dog.Dr. Burak ERMAN

Doç.Dr. Sabri ALTINTAŞ

Handa.

Binakkiman S. altintas

DATE OF APPROVAL



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I am also indepted to Turkish Army that gave me opportunity to get my entire graduate education in the Faculty of Engineering at Boğaziçi University.

FATIGUE CRACK GROWTH PREDICTION AND ANALYSIS

In this thesis, A computer program predicting the fatigue life of the specimens subjected to randomized black loading is prepared. A detailed description of fatigue crack growth and predictive models are reviewed.

The computer program AKYÜREK-I takes also the retardation phenomenon into consideration. The Wheeler and Willenborg retardation models are used.

In the program, center-crack panel, single and double edge cracked specimens, compact tension specimen, bend specimen and surface flawed plates are considered. A cycle-by-cycle integration scheme is used.

The results of some experiments are used to review the existing crack growth prediction models.

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YORULMA ÇATLAĞI İLERLEMESİ TAHMİN VE ANALİZİ

Bu tez çalışmasında, düzensiz blok yükleme tatbik edilen deney numunelerinin yorulma sürelerini tahmin eden bir bilgisayar programı hazırlanılmıştır. Ayrıca yorulma çatlak ilerlemesi ve yorulma süresi tahmin modellerini içeren geniş bir derleme yapılmıştır.

AKYÜREK-I bilgisayar programı çatlak ilerlemesindeki gecikme olgusunu da göz önüne almaktadır. Wheeler ve Willenborg gecikme modelleri kullanılmıştır.

Program merkez çatlaklı, tek ve çift kenar çatlaklı deney numuneleri, standard ufak çekme numunesi, bükme numunesi ve yüzey çatlaklı plakalara uygulanan yükleme durumlarını değerlendirmektedir. Çatlak ilerlemesi; yükleme dalgaları bir bir göz önüne alınarak, her bir yüklemenin neden olduğu ilerlemelerin toplamı olarak düşünülmüştür.

Bazı deney sonuçlarını kullanarak, programdaki çatlak ilerleme metotlarının değerlendirmeleri yapılmıştır.

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LIST OF SYMBOLS

a	Crack length
a/w	Crack length-the width of specimen ratio
A(f)	Some function of frequency
ai	Current crack length
ao	Crack length at the overload
ap	a _o +r _{po} (plastic zone size due to overload)
В	Thickness of the specimen
C	Material constant in-crack-growth equations.
C	Crack length in the major axis direction of the surface
	flaw
COD	Crack opening displacement
С _р	Retardation factor
da/dN	Crack growth rate
(da/dN)lin	.Linear crack growth rate
(da/dN)ret	.Retarded crack growth rate
ds	A segment of crack border
du	Strain energy change
E	Youngs modulus
G	Crack extension force
h	Step size in the simpson's integration formula
K	Stress intensity factor
ĸı	Stress intensity factor for mode-I
KIC	Plane-strain fracture toughness
K _{max.i}	Maximum stress intensity factor at the current cycle
K _{min.i}	Minimum stress intensity factor at the current cycle
K _{max.eff}	Max. effective stress intensity factor
K _{min.eff} -	Min. effective stress intensity factor

m	Walker's exponent, Wheeler's retardation exponent
M _K	Front-free surface correction
n	Material constant
N .	Cycles of load
p.c.f.	Plastic constraint factor
Pmax	Maximum applied load
P _{min}	Minimum applied load
Q	Flaw shape parameter
r ^ř pi	Plastic zone size at the current cycle
R	Cycle ratio (= K_{min}/K_{max})
r_p^{\star}	Plastic zone correction
S	Distance between the supports in bend specimen
U	Elber's stress intensity ratio ($= \Delta K \operatorname{eff} / \Delta K$)
Y	A polynomial in (a/w)
W	Width of the specimen
≪	Constant used in plastic zone size evaluation
Φ	Elliptic integral of the second kind
2	Crack opening displacement in y direction
20	Half the total separation of the wall at the origin
Get	Closure stress
Gmax	Maximum stress
Gmin .	Minimum stress
Ga	Stress amplitude (Cmax-Cmin)/2
Gres	Residual stress
Gy	Stress in the y direction
Gys	Yield strength
Gr	Stress range $(= C_{\max} - G_{\min})$
Δ Κ,	Stress intensity range

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 ΔK_{eff} ΔG Effective stress intensity range

Stress range (= G_{max} - G_{min})

I.INTRODUCTION

Fatigue of materials is a common problem that affects everything that moves, and even apparently stationary objects may fail in fatigue if they experience cyclically imposed forces or deformations. There are innumerable problem areas. Automobiles fatigue from rough roads, airplane wings flutter in flight, bridges deflect under each passing vehicle, ships are heaved by the waves, and nuclear reactors are damaged by cyclic temperature changes.

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The goal of this study is to prepare a computer program using the Wheeler and the Willenborg retardation models. This program predicts the cyclic life of the edge-cracked, center-cracked, double-edge-cracked, surface flawed, ASTN standard compact tension and bend specimens under randomized block loading. Since the computer program requires a high knowledge of fatigue crack propagation detailed review of that subject is given in Chapters II, III, and IV.

In reality materials experience variable amplitude load fatigue. Because of the difficulty in treating the problem in real conditions, research has been devoted to predict the variable amplitude behavior from the constant amplitude data.

The subject of linear crack growth is reviewed in Chapter II. Since stress intensity factor range is the governing parameter in crack propagation, stress intensity evaluations are presented. The crack tip plasticity is the basis of the Wheeler and the Willenborg retardation models, therefore it is also reviewed in Chapter II.

Many models have been developed to take into account the retardation phenomenon in variable amplitude fatigue. The best known four models; the Wheeler model-the Willenborg model-the crack closure model-the root mean square approach, are presented in Chapter III.

The models usually overestimate or underestimate the fatigue life. In orde to understand the reasons of that, the factors affecting crack propagation should be known. They are reviewed in Chapter IV. The computer program, AKYÜREK-I, is given in Chapter V. Sample calculations using AKYÜREK-I, their comparisons with the test data and the reasons of difference between calculations and the test data are presented in Chapter VI.

Chapter VII presents the conclusion of the study. Users manual for AKYÜREK-I is given in Appendix-D.

II. FATIGUE CRACK PROPAGATION

2.1. Fatigue Crack Growth Under Constant Amplitude Loading

Linear Elastic Fracture Mechanics (LEFM) regards the stress intensity factor as a sufficient parameter to describe the whole stress field at the tip of a crack. When there is a plastic zone at the crack tip, the stress intensity factor may still give a good indication of the stress environment of the crack tip, provided the size of the zone is small compared to the crack length. If two different cracks have the same stress environment, i.e. the same stress intensity factor, they behave in the same manner and show equal rates of growth (1)^{*}. The rate of fatigue crack propagation per cycle, da/dN, is governed by the stress intensity factor range, ΔK ;

 $da/dN = f(\Delta K) = f\{(G_{max} - G_{min})\sqrt{\pi a}\} = f\{2Ga\sqrt{\pi a}\}$ (2.1)

Where G max and G min are the maximum and minimum stresses in a cycle, and G a is the stress amplitude.

^{*} Parenthetical references placed superior to the line of text references to the bibliography.

Paris (2) and Paris, Gomez and Anderson (3) were first to point this out. Data obtained from specimens tested at various stress levels should all be on a single curve. Figure 2.1. shows a plot of data (4) obtained at three different stress levels, but with the minimum stress in a cycle always virtually zero (the cycle ratio $R = \zeta_{min}/\zeta_{max} = 0.05$).



, da/dN (um/cycle)

Figure 2.1. Relation between stress intensity factor and crack propagation rate (4).

The data in this figure indeed obey eq(2.1).

Equation (2.1) is sometimes assumed to be a simple power . function:

$$da/dN = C (\Delta K)^n$$
 (2.2)

in which C and n are supposed to be material constants. A double logarithmic plot of da/dN versus ΔK would then be a straight line. However, eq(2.2) does not fully represent reality. Actual data fall on a S-shaped curve, or on a line with varying slope (5,6), as shown in figures 2.1 and 2.2. At low ΔK values crack propagation is extremely slow. Conceivably there is a threshold value of ΔK below which there is no crack growth at all (e.g.7). Experimental

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verification of the existence of this threshold is difficult. A growing crack of some length has to be arrested by gradually decreasing ΔK until below the threshold, e.g. by decreasing the stress amplitude. The interpretation of the results presents often difficulties in view of history effects.



Figure 2.2. Crack growth rate versus stress intensity factor range (5).

According to the mechanism of fatigue crack growth, the amount of growth per cycle is closely related to the crack tip opening displacement. Therefore, attempts have been made (8,9) to correct late the crack propagation rate with the crack opening displacement. This leads to relations of the form

$$\frac{da}{dN} = \frac{C(\Delta K)^2}{E \zeta_{ys}} \quad \text{or} \quad \frac{da}{dN} = \frac{C}{\left(\frac{\Delta K}{E}\right)^2} \quad (2.3)$$

in which E is Young's modulus and ζ_{ys} is the cyclic yield stress. These equations are interesting, because crack propagation can be considered as a geometric consequence of crack tip opening (10, 11, 12). It has been shown (e.g.13) that data for a large variety of

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materials fall within one large scatter band when plotted on the basis of $\Delta K/E$ versus da/ dN, as suggested by the second expression in eqs (2.3). However, a mere glance at figures 2.1 and 2.2 shows how materials with virtually the same Young's modulus can have widely different crack propagation properties. This is probably due to the fact that many more parameters are involved than those occured in eq(2.3).

Many other equations have been proposed. They are analyzed in a concise paper by Pelloux (14). Further work to derive an equation with a sound physical basis is certainly needed; it must be anticipated that this final equation will be a complicated one if it is to have a general validity. For the technical problem of fatigue crack propagation the simple knowledge that da/dN is a function of the stress intensity factor will often be sufficient.

A fatigue cycle is defined by a frequency and two stress parameters. These can be the mean stress \mathcal{C}_m and the stress amplitude $\mathcal{C}_{\mathbf{R}}$, the minimum stress in a cycle $(\mathcal{C}_{\min} = \mathcal{C}_m - \mathcal{C}_{\mathbf{R}})$ and the maximum stress $(\mathcal{C}_{\max} = \mathcal{C}_m + \mathcal{C}_{\mathbf{R}})$, or other combinations of two of these four parameters. As long as the cycle ratio $(\mathbf{R} = \mathcal{C}_{\min} / \mathcal{C}_{\max})$ equals zero, one can speak unambiguously about the stress intensity factor of the fatigue cycle, since $\mathcal{C}_{\max} = 2\mathcal{C}_{\mathbf{R}} = \Delta \mathcal{C}$. The hypothesis that the rate of crack propagation is a function of stress intensity factor presents no difficulties. When $\mathbf{R} \neq 0$ the range of the stress intensity $\Delta \mathbf{K} = 2\mathcal{L}_{\mathbf{R}}\sqrt{\pi \mathbf{a}}$ is an insufficient description of the stress environment of the crack tip. The question arises whether da/dN will now be a function of $\Delta \mathbf{K}$, or of the maximum stress intensity in a cycle $(\mathbf{K}_{\max} = \mathcal{L}_{\max}\sqrt{\pi \mathbf{a}})$ or of both.

It appears (15,16) that the rate of crack propagation is a function of both ΔK and K_{max} . This can be appreciated from figure 2.3. It can be concluded that

$$da/dN = f_1(\Delta K, R) = f_2(K_{max}, R) = f_3(\Delta K, K_{max})$$
 (2.4)

Several investigators have tried to establish empirical relations which attempt to incorporate the effect of the cycle ratio such that all data could be condensed to a single curve. Broekk and Schijve (15) proposed a complicated relation, but also the following more simple one:

$$da/dN = C K_{max}^{2} \Delta K \qquad (2.5)$$

A similar equation was given by Erdoğan (16). Walker (17,18) used the more general equation

$$da/dN = C K_{max}^{m} \Delta K^{n}$$
 (2.6)

which he modified by introducing an effective $\overline{\Delta K}$, yielding

₩ = S_{max} (l-R)**A**a 80 Kmax (kg/mm 70 60 50 40 30 20 30 = c.4 V 20 R=0.5 A 15 0. I 0.01 ea, mm/kc 99

Figure 2.3.

Effect of cycle ratio on the relation between crack growth rate and stress intensity factor (15),2024-T3alloy.





(2.7)

Forman et al. (19) argued that da/dN should become infinite when the crack reaches a critical crack size, i.e. when K_{max} reaches K_{1C} . They arrived at

$$\frac{da}{dN} = \frac{C \Delta K^{n}}{(1-R)K_{1C} - \Delta K} = \frac{C \Delta K^{n}}{(1-R)(K_{1C} - K_{max})}$$
(2.8)

which can be rearranged to give:

$$\frac{da}{dN} = \frac{C \Delta K^{n} K_{max}}{K_{1C} - K_{max}}$$
(2.9)

The differences among these expressions are not large, and none of them has a general applicability. Each one may be found reasonably satisfactory in a limited region or for limited sets of data.

The question arises whether eq(2.4) still holds for $R \lt 0$, i.e. when the stresses go into compression. A crack is not a stress raiser in compression and the expressions for K lose their meaning. This suggests that

$$da/dN = f(K_{max})$$
 for $R \lt 0$ (2.10)

There have been many arguments about the validity of eq(2.10). The data (20) plotted in figure 2.4 seem to support the equation. A crack will not always close exactly at the moment the stress reverses from tension into compression.



Figure 2.4. Crack propagation rate at negative cycle ratios (20). 7075-T6 Al-alloy.

The moment of closure depends upon the magnitude of the crack tip opening attained in the tension part of the cycle, and upon the plastic deformation properties of the material (20). Therefore, eq(2.10) should probably be modified to

 $\frac{da}{dN} = f_{1} (\Delta K, K_{max}) = f_{2}(K_{max}, R) \text{ for } R \neq \delta$ $\frac{da}{dN} = f_{3} (K_{max}) \text{ for } R \neq \delta$

= f_4 (material properties) \approx o. (2.11)

2.2. Stress intensity formulations and the effect of finite size.

Cracks in a plate of finite size are of great practical interest, but for these cases no closed form solutions are available. The problems are difficult because of the boundary conditions. An approximate solution can be obtained for a strip of finite width loaded in tension and containing an edge crack or a central crack (22).

First consider an infinite sheet with an infinite row of evenly spaced collinear cracks as depicted in figure 2.5. Solutions for this case were given by Westergard (23), Irwin (24) and Koiter (25). The result is:



Figure 2.5. Infinite plate with collinear cracks.

$$K_{I} = G \sqrt{\pi a} \left(\frac{W}{\pi a} \tan \frac{\pi a}{W} \right)^{\prime 2}$$
(2.12)

If the plate is cut along the lines AB and CD one obtains a strip of finite width w, containing **a** central crack 2^a . It is likely that the solution of eq (2.12) is approximately valid for the strip. In the case of collinear cracks a strip of width w bears stresses (note that shear stresses are zero because of symmetry) along its edges AB and CD (figure 2.6), where as the edges of a plate of finite size are stress free. Supposedly, the stresses parallel to the crack do not contribute much to K and consequently eq (2.12) can be used as an approximate solution for the strip of finite size.

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It appears that



Figure 2.6. Stresses on the edges of strip cut from infinite plate with collinear cracks.

eq (2.12) reduces to $K_{I} = G \sqrt{\pi a}$ if $(^{a}/w)$ approaches zero. This means that the finite strip behaves as an infinite plate if the cracks are small.

Isida (26) has developed mapping functions to derive stress concentration factors. These can be used (27) to compute stress intensity factors for finite plates to any degree of accuracy. Usually the result is presented as:

$$X = Y \mathcal{G} \sqrt{a}$$
(2.13)

where Y is a polynomial in a/w. The factor $\sqrt{\pi}$ is often incorporated in Y, sometimes it is not. Fedderson (28) discovered that the solution of Isida is very closely appoximated by $\sqrt{\sec \pi a/w}$. Therefore a convenient formula for the stress intensity factor for a strip in tension is

$$K_{I} = G \sqrt{\pi a} \sqrt{\sec \pi a / w}$$
(2.14)

A comparison of a finite width correction factors of Irwin, Iside and Fedderson is made in Figure 2.7.



Figure 2.7. Finite width corrections for center cracked plate.

Cutting the plate with collinear cracks (figure 2.5) along EF and CD, one similarly arrives at a strip with an edge crack. Analogous to the central crack problem the solution of eq(2.12) can be used as an approximation for the edge crack. Again K reduces to $K = C\sqrt{\pi a}$ for small ^a/w. However, the stresses acting on the edge EF tend to slightly close the crack. Absence of these stresses in the strip of finite size results in a somewhat larger displacement of the crack edges. Consequently K is somewhat higher due to these free edges. The correction factor is in the order of 12 percent (27). Thus, for a small edge crack K is given by

$$K_{T} = 1.12 \, 6 \sqrt{\pi a}$$
 (2.15)

Stress intensity factors and the finite size polynomials for a number of practical configurations are collected in table 2.1. TABLE 2.1.

K expressions for some practical geometries. 24 $K_{I} = G \sqrt{\pi a} (\sec \pi a/w)^{1/2}$ $K_T = 1.12 \ \sqrt{\pi a} \ (\text{small } a/w)$ or $K_{I} = Y C \sqrt{a}$ with $Y = 1.99 - 0.41 \frac{a}{w} + 18.7 (\frac{a}{w})^{2} - 38.8 (\frac{a}{w})^{3} + 53.85 (\frac{a}{w})^{4}$ $(1.99 = 1.12\sqrt{\pi})$ $K_{\tau} = 1.12 G \sqrt{\pi a} (\text{small } a/w)$ or $K_{I} = Y C \sqrt{a}$ with $Y = 1.99 + 0.76 - \frac{a}{w} - 8.48 (\frac{a}{w})^{2} + 27.36 (\frac{a}{w})^{3}$ $(1.99 = 1.12 \sqrt{\pi})$ ASTM BEND SPECIMEN $\ddot{K}_{I} = \frac{PS}{RW^{3/2}} \left[2.9 \left(\frac{a}{W}\right)^{-1/2} 4.6 \left(\frac{a}{W}\right)^{3/2} + 21.8 \left(\frac{a}{W}\right)^{-37.6} \left(\frac{a}{W}\right)^{3/2} + 38.7 \right]$ Thickness: B ASTM COMPACT TENSION SPECIMEN $K_{I} = \frac{P}{Bw^{\nu_{2}}} \left[29.6(\frac{a}{w})^{\nu_{2}} - 185.5(\frac{a^{3/2}}{w}) + 655.7(\frac{a^{5/2}}{w}) - 1017(\frac{a}{w})^{\frac{3}{2}} + 63.9(\frac{a^{5/2}}{w})^{\frac{3}{2}} +$ Thickness:B $K_{\text{Imax}} = 1.12 \frac{G}{\Phi} \sqrt{\pi a}$ $K_{\text{Imin}} = 1.12 \frac{c}{\phi} \sqrt{\pi a^2/c}$

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2.3. Stress Intensity Evaluations in Semi-elliptical Surface Flaws,

Natural cracks occuring in practice are often initiated at corners and edges. They tend to grow inwards and assume a quarterelliptical or semi-elliptical shape. The application of fracture mechanics to these "corner cracks" (quarter elliptical) and to "surface flows" or "part-through" cracks (semi-elliptical) requires knowledge of the stress intensity factor for a crack with a curved front. Because of its technical significance this problem has received ample attention in the literature (29-40). A widely used approximate solution is discussed in the fellowing paragraphs.

Sneddon (29) treated the problem of a circular internal crack of radius a (penny-shaped crack) embedded in an infinite solid subjected to uniform tension (figure 2.8).



Figure 2.8. Embedded penny-shaped crack. He arrived at:

$$K_{I} = \frac{2}{\pi} G \sqrt{\pi a}$$
(2.16)

A solution for an embedded elliptical flaw not being available, Irwin (24) derived a useful expression on the basis of the stress field around an ellipsoidal cavity as derived by Green and Sneddon (30). The displacements found from the latter solution were rela-

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ted (29) to the stress intensity factor in the same way as in the case of through-the-thickness cracks.

Studies of the flat elliptical crack problem by Green and Sneddon (30) provide principally the information that tension normal to the crack produces an ellipsoidal crack opening shape. Assume the crack lies in the X-Z plane with its major dimension 2**G** along the Z-axis and its minor dimension 2a along the X-axis in such a way that border points X_1, Z_1 of the crack correspond to

$$\frac{x_1^2}{a^2} + \frac{z_1^2}{c^2} = 1$$
 (2.17)

If the crack opening displacements in the y-direction are represented by ? then the Green-Sneddon result may be expressed as

$$? = ? \circ (1 - \frac{x^2}{a^2} - \frac{z^2}{c^2})^{1/2}$$
 (2.18)

where ?o is half the total separation of the walls of the crack at the origin.



Figure 2.9. Part-through crack in a plate showing dimensions a and c of crack and plate thickness, B.

The desired expressions for \mathcal{G} and K will be derived from eq (2.18) rather than from stress equations for two reasons: (a) The stress relations given by the Green-Sneddon paper are not in a form convenient for the calculation of \mathcal{C}_y near the border of the crack; (b) The author wishes to establish the point that the general shape of the crack opening provides sufficient information to determine \mathcal{G} and K.

The procedure to be followed consists first in a discussion of the variation around the border of the crack of the elastic opening displacements of the crack. It will be noted that deviations of the stress state from plane strain become negligible in the limit of small separations from the crack border. Plane strain relations for \mathbf{G} and K associated with crack-opening displacement then permit the desired calculations.

The position variables X_1 and Z_1 which lie on the crack borders may be represented in parametric form by

$$Z_1 = c \cos \phi \qquad (2.20)$$

A change dø then corresponds to a segment ds of crack border given by

$$ds = (a^2 \cos^2 \phi + c^2 \sin^2 \phi) d\phi \qquad (2.21)$$

At a small normal separation r (inward from the crack border) straight forward algebraic steps from eq(2.18) lead to the expression

$$q^{2} = \sqrt{2^{2} \frac{2r}{ac}} \left(a^{2} \cos^{2} \phi + c^{2} \sin^{2} \phi\right)^{1/2}$$
(2.22)

The variation of displacement ? at fixed distance r from the crack border as a function of ϕ will be considered in terms of the variation with ϕ of

$$e = i^2 / 2r$$
 (2.23)

Thus

$$e^{2} = \frac{2 o^{2}}{ac} (a^{2} \cos^{2} \phi + c^{2} \sin^{2} \phi)^{1/2}$$
(2.24)

Differentiating with respect to ϕ and using eq(2.21) one finds

$$\frac{\mathrm{d}\,\mathbf{e}}{\mathrm{d}s} = \left(\frac{2\mathrm{o}^2}{\mathrm{ac}}\right) \frac{(\mathrm{c}^2 - \mathrm{a}^2)\,\sin\,\phi\,\cos\,\phi}{\mathrm{a}^2\,\cos^2\phi + \mathrm{c}^2\,\sin^2\phi} \tag{2.25}$$

At the value of ϕ where $d\rho$ /ds is largest the fractional change of ρ across the segment ds is

$$d e /e = \frac{(c^2 - a^2) (a^2 + c^2)^{1/2} ds}{2 \sqrt{2} a^2 c^2}$$
(2.26)

putting c = 2a for specific illustration, the value of $d\rho/\rho$ becomes

$$de /e = 0.6 ds/a$$
 (2.27)

The quantity ρ can be interpreted as the root radius of the elastic crack opening along the crack border. As a reference point approaches the crack border, ρ becomes the principal length factor associated with the crack. If we know a solution for stresses and strains valid along a crack-border length δ s large enough so that the dimension ρ is negligible in comparison, then such a solution can be used to supply an expression for γ for substitution into eq(2.22)

(2.31)

In crack stress-field analysis problems ρ is always equal to $(\mathcal{C} / \mathbb{E}^2)$ times a dimension comparable to the crack size. Bearing in mind that linear elacticity analysis regards \mathcal{C} / \mathbb{E} as an infinitesimal in comparison to unity, we can for example, choose.

$$\delta S = \left(\frac{E}{G}\right) Q \qquad (2.28)$$

or, say,

$$\delta S = \left(\frac{G}{E}\right) a \tag{2.29}$$

Such a crack-border-length segment is obviously short compared to crack size and long compared to e, substituting δ s from eq(2.29) for ds in eq(2.27) gives

$$d\rho = 0.6 (G/E)$$
 (2.30)

which can be regarded as meaning e changes only by an infinitesimal fraction across the border length segment δ s. The changes of displacements and stresses near the crack border must possess a similar degree of constancy relative to a coordinate parallel to the crack border. Since the problem is symmetrical about the plane containing the crack, a plane strain stress field fits the conditions of the problem in local crack-border regions comparable to dimensions to δ s.

The three crack-border stress fields for which the displacements are independent of the co-ordinate prallel to the border are discussed in reference (41). The first of these, opening mode

 $? = \frac{2(1 - v^2)}{r} (2r)^{1/2} K$

$$G = \frac{\pi}{E} (1 - \gamma^2) K^2$$
 (2.32)

Inserting γ^2 from eq(2.31) to \mathcal{C} , a and c from reference (31) or find this relation by computing the strain energy change for a small expansion of the crack boundary. The latter is not difficult. If the ellipse is expanded by adding fa to a and fc to c where f is very small, the normal outward displacement, r, of any point on the elliptical boundary is

$$r = \frac{acf}{(a^2 \cos^2 \phi + c^2 \sin^2 \phi)^2} / 2$$
 (2.33)

Computation of the strain energy change dU from

$$du = 4 \int_{0}^{\pi/2} \int \frac{ds}{d\phi} d\phi$$
 (2.34)

leads to

$$du = \frac{\pi \text{Ecf } \mathbf{l}_0^2 \mathbf{\Phi}}{(1 - \nu^2)}$$
(2.35)

where ϕ is the elliptic integral

$$\bar{\Phi} = \int_{0}^{\pi/2} \left[\sin^{2} \phi + \left(\frac{a}{c} \right)^{2} \cos^{2} \phi \right]^{\frac{1}{2}} d\phi$$
(2.36)

we also know the strain energy change is one half of C times the change in crack volume. From this

$$du = 2\pi Gacf_{o}$$
 (2.37)

The value of ? found by equating the two relations for du equations (2+35) and (2+37) may be substituted into (2+32) with the result that

$$\mathcal{G} = \frac{\pi (1 - \nu^2) a^2}{E \phi^2} \left(\frac{a}{c}\right) \left(a^2 \cos^2 \phi + c^2 \sin^2 \phi\right)^{1/2} \quad (2.38)$$

By inspection of eq(2.38) one observes that G is greatest where the crack-boundary intersects the minor axis. Thus with increase of tension on a flat elliptical crack, the crack extension should, barring anisotropy, tend to produce a circular crack boundary shape.

The function ϕ increases with $(\frac{a}{-})$. When a is infinitesimal compared to c, ϕ is unity. When crack shape is circular ϕ is $\pi/2$. Other values of ϕ are easily found from published tables of elliptical integrals. Taking $\phi = \pi/2$ it is readily seen that the values of unity and $\pi/2$ for ϕ correspond, respectively, to the Griffith equation

$$G = \frac{\pi G_{\rm a}}{N_{\rm E}} (1 - \gamma^2)$$
 (2.39)

and to the energy-release rate for the penny-shaped crack

$$G = \frac{4 G_{\rm B}^2}{\pi E} (1 - \gamma^2)$$
 (2.40)

The results of Irwin's analysis is

- 1-

$$K_{I} = \frac{\sqrt{\pi a}}{\Phi} \left(\sin^{2} \phi + \frac{a^{2}}{c^{2}} \cos^{2} \phi \right)^{1/4}$$
(2.41)

in which ϕ is an elliptical integral of the second kind, given by

$$\oint = \int \left[1 - \frac{c^2 - a^2}{c^2} \sin^2 \phi \right]^{1/2} d\phi$$
 (2.42)

where a and c are as defined in figure 2.10. If a = c



Figure 2.10. Elliptical crack

eq (2.41) reduces to eq(2.16), as should be the case. Values for \oint can be found in mathematical tables or in a graph as in figure 2.11. It is possible to develop a series expansion for \oint

$$\Phi = \frac{\pi}{2} \left\{ 1 - \frac{1}{4} \frac{c^2 - a^2}{c^2} - \frac{3}{64} \left(\frac{c^2 - a^2}{c^2} \right)^2 - \dots \right\}$$
(2143)

Even for a ratio a/c approaching zero the third term contributes only about 5 per cent and therefore it can be neglegted in most cases, yielding



Figure 2.11. Surface flaw parameter.

and also:

$$K_{I} = \frac{6\sqrt{\pi a}}{\frac{3\pi}{8} + \frac{\pi}{8} \frac{a^{2}}{c^{2}}} (\sin^{2}\phi + \frac{a^{2}}{c^{2}} \cos^{2}\phi)^{1}/4$$
(2.

(2.44)

45)

With only slight modifications eqs(2.41) and (2.45) can be applied to semielliptical surface flaws and to a quarter-elliptical corner cracks. Therefore the equations are of great practical interest. It turns out that K_{I} varies along the crack front. At the end of the minor axis ($\phi = \pi/2$) the stress intensity is the largest. At the end of major axis ($\phi = \phi$) it is the lowest. Therefore;

$$K_{I}(\phi = \pi/2) = \frac{G\sqrt{\pi a}}{\Phi}$$
; $K_{I}(\phi = 0) = \frac{G\sqrt{\pi a^{2}/c}}{\Phi}$ (2.46)

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Usually a number of correction factors are applied to these Kexpressions. A surface flaw is comparable to an edge crack, and it was argued that this requires a correction factor of about 12 percen to K. This is called the back free-surface correction. Also, a plastic zone correction is often applied to take account of the fact that plastic deformation takes place at the crack tip. This plastic deformation makes the crack behave as if it were slightly longer than its physical size. Because of this, the plastic zone correction r_{p}^{*} is a correction to the crack size;

$$K_{I}=1.12 \frac{\sigma}{\Phi} \sqrt{\pi} (a \pm r_{p}^{*}) (\sin^{2} \sigma + \frac{a^{2}}{c^{2}} \cos^{2} \sigma)^{1/4}$$
 (2.47)

By taking

$$r_{p}^{*} = \frac{\frac{K_{1}}{\sqrt{2}}}{4\pi\sqrt{2}} \qquad (C_{ys} \text{ is the yield stress}) \qquad (2.48)$$

the resulting expression for K is :

$$K_{I} = \frac{1.12 \, 6 \sqrt{\pi a}}{\sqrt{\phi^{2} - 0.212 \, 6^{2} / {G_{ys}}^{2}}} (\sin^{2} \phi + \frac{a^{2}}{c^{2}} (\cos^{2} \phi)^{1/4} (2.49)$$

The maximum stress intensity is: $K_{\text{Imax}} = 1.12 6 \sqrt{\pi a/Q}$ (2.50)
The quantity $Q = \frac{\Phi^2}{\Phi^2} - 0.212 \frac{G^2}{G_{ys}}^2$ is called the flaw shape parameter. Values for Q are presented graphically in figure 2.11 for various values of the ratio $\frac{G}{G_{ys}}$.

Finally, a correction is often necessary to account for the proximity of the free surface in front of the crack. For this front free-surface correction, use can be made of the tangent formula of eq(2.12). It is preferable, however, to use the front free-surface correction due to Kobayashi et al. (31), which is given in graphical form in figure 2.12. The resulting maximum stress intensity for a surface flaw becomes:

$$K_{\rm Imax} = 1.12 M_{\rm K} \, G \sqrt{\pi a/Q}$$
 (2.51)

where M_{K} is the front free-surface correction of figure 2.12.



Figure 2.12. Kobayashi correction (M_K) for proximity of front freesurface.

For the case where a semi-elliptical flaw extends deep into the material, the back free-surface correction should be decreased from 1.12 to unity. For the case of a quarter-elliptical crack having two free surfaces, the back free-surface correction should

be applied twice. However, it appears that this is a slight overcorrection. Therefore the back free-surface correction for a corner crack is usually taken as 1.2.

The previous equations for surface flaws were obtained indirectly (24) from the solution for an embedded elliptical cavity. Rice (32) and Rice and Levy (33) have directly analysed the problem of a surface crack. Their finelequations can be treated to give numerical data for K, which are particularly useful because the bending case was also solved. It turns out that for shallow flaws (2c/B large) the stress intensity factor approaches the value for an edge crack ($2c \rightarrow \infty$). The same result is obtained from eq(2.51) since Q = 1 for a/2c = 0. Stress intensity factors for surface cracks in bending were also calculated by Grandt and Sinclair (34). The foregoing discussion serves as an illustration of the variation of the stress intensity factor along the crack front of a surface flaw. Information about stress intensity factors of elliptical cracks can be found elsewhere (31-40).

2.4. The Crack Tip Plastic zone

2.4.1. Irwin Plastic Zone Correction

According to the elastic stress field solutions a stress singularity exists at the tip of an elastic crack. In practice, materials (especially metals) tend to exhibit a yield stress, above which they deform plastically. This means that there is always a region around the tip of a crack in a metal, where plastic deformation occurs, and hence a stress singularity cannot exist. The plastic region is known as the crack tip plastic zone. A rough estimate of the size of the plastic zone, whether in plane strain

or plane stress, is simple to make. To start with, the considerations in this section are limited to plane stress.



Figure 2.13. First estimate of plastic zone size.

Figure 2.13 shows the magnitude of the stress ζ_y in the plane $\theta = 0$. Until a distance r_p^* from the crack tip the stress is higher than the yield stress ζ_{ys} . To a first approximation this distance r_p^* is the size of plastic zone. By substituting ζ_{ys} in the equation for ζ_y the distance r_p^* can be calculated:

$$G_{y} = \frac{K_{I}}{\sqrt{2\pi r_{p}^{*}}} = G_{ys} \text{ or } r_{p}^{*} = \frac{K_{I}^{2}}{2\pi G_{ys}^{2}} = \frac{G_{\pi a}^{2}}{2\pi G_{ys}^{2}}$$
 (2.52)

It is quite clear that the actual plastic zone size must be larger than r_p^* : the load represented by the shaded area in figure 2.13 must still be carried through. This can be achieved if the material immediately ahead of the plastic zone carries some more stress which will bring this material above the yield stress.

Irwin (42) has argued that the occurence of plasticity makes the crack behave as if it were longer than its physical size. As a result of crack tip placticity the displacements are larger and the stiffness is lower than in the elastic case. In other words, the plate behaves as if it contained a crack of somewhat larger size. The effective crack size, aeff, is equal to a $\pm \delta$, the physical crack size plus a correction δ . An expression for can easily be derived.

In figure 2.14 the physical crack of size a is replaced by a longer crack of size a $\pm \delta$, and the elastic stress distribution (G_y) at the tip of the effective crack is given. The stress at the tip of the effective crack is again limited to the yield stress G_{ys} . Similarly, the stress acting on the part δ in front of the physical crack is equal to the yield stress. Consequently, δ must be large enough to carry the load that is lost by cutting the area A (figure 2.14) from the elastic



Figure 2.14. Second estimate of plastic zone size.

stress distribution. Hence, area A is equal to area B. The distance λ in figure 2.14 follows from:

$$G_{ys} = \frac{K}{\sqrt{2\pi\lambda}} = G \sqrt{\frac{a+\delta}{2\lambda}} \quad \text{or } \lambda = \frac{G^2(a+\delta)}{2Gys} \approx r_p^*$$
 (2.53)

Since δ is small with respect to the crack size it can be neglected, and it follows that $\lambda \approx r_p^*$ as in eq(2.52). The area B is

equal to $\mathcal{O}_{ys} \cdot \delta$; hence, the requirement B = A yields:

$$\delta \cdot \mathcal{C}_{ys} = \left[\int \mathcal{O} \sqrt{\frac{a-\delta}{2r}} \, dr \right] - \mathcal{O}_{ys} \lambda.$$
(2.54)

Neglecting δ as compared to a and using eq(2.53) it follows that

$$(\delta + r_p^*) G_{ys} = G \sqrt{2ar_p^*}$$
 or $(\delta + r_p^*)^2 = \frac{2Ga}{Gys} r_p^* = 4 r_p^{*2}$ (2.55)

Hence, it turns out that:

$$\delta = r_p^* \text{ and } r_p = \lambda + \delta = 2r_p^*.$$
 (2.56)

The size of the plastic zone r_p is found to be twice as large as the first estimate, r_p^* .

Since $\delta = r_p^*$ it follows that the crack behaves as if its length were a + r_p^* . The quantity r_p^* is known as Irwin's plastic zone correction. Assuming for the time being that the plastic zone has a circular shape, the situation can be represented as in figure 2.15, where the effective crack



Figure 2.15. Irwin's plastic zone correction.

extends to the centre of the plastic zone. If the plastic zone correction is applied consistently a correction to K is also

necessary:

$$K = C G \sqrt{\pi (a + r_p^*)} = C G \sqrt{\pi (a + \frac{K^2}{2\pi G_{ys}^2})}$$

(2.57)

The use of eq(2.57) presents difficulties because K has to be determined by following an iteration procedure. The latter can be avoided if one takes $K = C \sigma \sqrt{\pi a}$ for calculating r_p^* and then determines the corrected K from eq(2.57). Conversely, for a given K one can find the uncorrected stress from $\sigma = K/\sqrt{\pi a}$, which allows determination of r_p^* . The corrected stress then follows from $\sigma = K/\sqrt{\pi (a + r_p^*)}$. In practice the plastic zone correction is seldom applied to K. The plastic zone correction of eq(2.52) is not suitable in plane strain (see sect.2.4.3.).

2.4.2. The Dugdale Approach:

A different approach to finding the extend of the plastic zone was followed by Dugdale (43,44) and (in a slightly different way) by Barenblatt (45). The procedure yields similar results as an analysis by means of a continuous distribution of dislocations (46,47).

Dugdale also considers an effective crack which is longer than the physical crack as in figure 2.16 a. The crack edges, ρ , in front of the physical crack carry the yield stress σ_{ys} , tending to close the crack. (The part ρ is not really cracked; the material can still bear the yield stress). The size of ρ is chosen such that the stress singularity disappears: K should be zero. This means that the stress intensity K₆



Figure 2.16. Dugdale approach; a. Dugdale crack, b. Wedge forces. due to the uniform stress \checkmark has to be compensated by the stress

intensity Ke due to the wedge forces G_{ys} :

$$K_{\sigma} = -K_{\rho} \tag{2.58}$$

The requirement (2.58) permits determination of ρ in the following manner. The stress intensity due to wedge forces p in figure 2.16 b is given as:

$$K_{A} = \frac{p}{\sqrt{\pi a}} \sqrt{\frac{a+x}{a+x}}$$
 and $K_{g} = \frac{p}{\sqrt{\pi a}} \sqrt{\frac{a-x}{a+x}}$ (2.59)

If the wedge forces are distributed from S to the crack tip (as in the Dugdale case) the stress intensity becomes:

$$K = \frac{p}{\sqrt{\pi a}} \int_{a-x}^{a} \left\{ \sqrt{\frac{a+x}{a-x}} + \sqrt{\frac{a-x}{a+x}} \right\} dx = 2p \sqrt{\frac{a}{\pi}} \int_{a}^{a} \frac{dx}{\sqrt{a^2 - x^2}}$$
(2.60)

The integration can be carried out by substituting $x=a\cos\phi$. The result is:

$$K = 2p \sqrt{\frac{a}{\pi}} \arccos \frac{S}{a}$$
 (2.61)

Applying this result to the Dugdale crack in figure 2.16 a the integral has to be taken from s = a to $a + \rho$. Hence a has to be substituted for s and $a + \rho$ for a in eq(2.61), while $p = G_{ys}$. Thus

$$K_{\varrho} = 2G_{y,g}\sqrt{\frac{a+\rho}{\pi}} \quad \arccos \frac{a}{a+\rho}$$
 (2.62)

According to eq(2.58) this stress intensity should be equal to $K_{\mathcal{G}}$, where the latter is $K_{\mathcal{G}} = \mathcal{G}\sqrt{\pi(a+\varrho)}$. Then it follows that ρ can be determined from eq(2.58) as

$$\frac{a}{a+e} = \frac{\cos \frac{\pi G}{2G_{ys}}}{(2.63)}$$

Neglecting the higher order terms in the series developement of the cosine, ρ is found as:

$$\varrho = \frac{\pi^2 G^2 a}{8 G_{ys}^2} = \frac{\pi \kappa^2}{8 G_{ys}^2}$$
(2.64)

This result can be compared with $r_p = 2r_p^* = K^2/\pi \zeta_{ys}^2$ as derived in the previous section. Apparently, the two expressions are almost identical. For high values of ζ/ζ_{ys} eq(2.63) has to be used instead of (2.64), and the differences with the Irwin plastic zone size become larger.

Duffy et al. (49) used eq(2.63) as a basis for a plastic zone correction. By taking $\rho = r_p^*$ it follows that a + $r_p^* =$ asec $\pi C/2 C_{ys}$ and $K_I = C \sqrt{\pi} \operatorname{asec} \pi C/2 C_{ys}$. Several other plastic zone corrections have been proposed. Correcting for plasticity is not necessary in the event that linear elastic fracture mechanics apply, i.e. when the plastic zone is small compared to the crack size. If the plastic zone is larger with respect to the crack, the application of a plastic zone correction is doubtful, because of the validity of the expressions for K, which are based on elastic solutions.

2.4.3. Plastic Constraint Factor.

The plane strain plastic zone is significantly smaller than the plane stress plastic zone. This is a result of the fact that the effective yield stress in plane strain is larger than the uniaxial yield stress. The maximum stress in the plane strain plastic zone can be as high as three times the uniaxial yield stress. The ratio of the maximum stress to the yield stress is called the plastic constraint factor (p.c.f.):

$$p.c.f = \frac{c_{max}}{c_{vs}}$$

The quantity (p.c.f.) times \mathcal{C}_{ys} can be considered as an effective yield stress.

The p.c.f. for the plane strain crack problem can be estimated as follows. By taking $\zeta_2 = n \zeta_1$ and $\zeta_3 = m \zeta_1$ the Von Mises yield criterion can be rewritten as:

$$[(1-n)^{2} + (n-m)^{2} + (1-m)^{2}]C_{1}^{2} = 2C_{ys}^{2}$$
(2.66)

which can be rearranged to:

p.c.f. =
$$\frac{G_1}{G_{ys}} = (1 - n - m + n^2 + m^2 - mn)^{-1/2}$$
 (2.67)

Eq (2.67) enables calculation of the p.c.f. at any location of the crack tip region. From the stress field equations it follows that $n = (1 - \sin e/2)/(1 + \sin e/2)$ and $m = 2 \nu/(1 + \sin e/2)$.

(2.65)

For the plane e = o it turns out that n = 1 and $m = 2 \forall$, by taking $\forall = \frac{1}{3}$ the plastic constraint factor is according to (2.67): p.c.f. = 3. Similar results are obtained by application of other yield criteria. In the case of plane stress n = 1 and m = o, which gives the estimate: p.c.f. = 1.

Apparently the normal stress ζ_y on the e = o plane in plane strain can be as high as three times the yield stress. During plase tic deformation the crack tip blunts. Since a stress perpendicular to a free surface cannot exist, it follows that ζ_x must tend to zero at the very crack tip. In that case $\zeta_2 = o$, i.e. there is a state of plane stress. Consequently p.c.f. must drop to 1 and stress at the crack tip does



Figure 2.17. Approximate stress distribution in plane stress and plane strain. a.Plane stress; b.Plane strain.

not exceed the yield stress. The resulting stress distribution are shown in figure 2.17. In the plane strain case the stress rises quickly from C_{ys} at the very crack tip to 3 C_{ys} at a short distance from the crack. This is confirmed by finite elemen calculations(50). Stress and strain distributions in the plastic zone measured as well as calculated, can be found in the literatur (51-57), but a general elastic-plastic analysis of the crack problem is not yet available.

Figure 2.17 again shows that the plastic zone in the y = oplane in the case of plane stress is nine times larger than for plane strain. Knowledge of the plastic constraint factor enables derivation of a plastic zone correction factor for plane strain in a similar way as in the above equation. If the effective yield stress in plane strain is $3c_{ys}$, the plastic zone correction of eq(2.52) becomes:

$$r_{p}^{*} = \frac{K_{I}^{2}}{2\pi (3\zeta_{ys})^{2}} = \frac{K_{I}^{2}}{18\pi \zeta_{ys}^{2}}$$
(2.68)

In a practical case, plane strain does not exist at the specimen surface. As a consequence the average plastic constraint factor is much lower than 3. Irwin (42) uses a p.c.f. of $\sqrt{2\sqrt{2}} = 1.68$, which modifies eq(2.68) into:

$$\mathbf{r}_{p}^{*} = \frac{K_{I}^{2}}{2\pi (1.68 \, G_{ys})^{2}} \approx \frac{1}{6 \, \pi} \, \frac{K_{I}^{2}}{G_{ys}^{2}} \tag{2.69}$$

This plastic zone correction is only one third of the plane stress correction. Experimentally determined p.c.f.'s (e.g. 58) are mostly between 1.5 and 2, which confirms the usefulness of eq(2.69).

III. RETARDATION MODELS

A fatigue cycle preceded by a load of higher magnitude produces less crack propagation than it does in the absence of the higher preload. This retardation phenomenon is usually attributed to a combination of compressive residual stresses and crack closure due to residual stresses (59).

The overload has introduced a large plastic zone as is shown in figure 3.1. The material in this zone is streched to a permanent deformation, but after unloading it still has to fit in the surrounding elastic material. The elastic material resumes its original size, but the material in the plastic zone does not. The plastic zone is too large for its elastic surroundings if the latter contract upon load release. Then the

after overload

Figure 3.1. Residual compressive stresses at crack tip as a result of overload.

elastic material has to make it fit. Consequently, the surrounding elastic material will exert compressive stresses on the plastically deformed material at the crack tip. The resulting residual stress system is depicted diagrammatically in figure 3.1. As soon as the crack has grown through the area of residual stresses, the original crack propagation curve will be resumed again. The residual compressive stresses tend to close the crack tip over some distance (60). Subsequent cycling can cause crack growth only if the residual stresses are overcome to a degree that the crack tip is opened again. This explains the low growth rate after the overload.

At least five models have been proposed (61-65) to treat retardation in a quantitative fashion. None of the models has a solid physical basis, and most are semiempirical, and contain one or more constants to be derived from variable-amplitude crack growth experiments.

3.1. The Wheeler Model

Wheeler (61) introduces a crack growth reduction factor, c_{p} ,

$$\frac{da}{dN} = C_p f(\Delta K)$$
(3.1)

where $f(\Delta K)$ is the usual crack growth function. The retardation factor, C_{p} , is given as (see figure 3.2):

$$C_p = (\frac{r_{pi}}{ao + r_{po} - ai})^m$$

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(3.2)

where

r_{pi}= current plastic zone in the ith cycle under consideration ai = current crack size

 r_{po} = size of the plastic zone generated by a previous overloa ao = crack size at which that overload occured m = empirical constant (retardation exponent)





The size of the plastic zone is:

$$\mathbf{r}_{\mathrm{pi}} = \frac{\mathbf{K}_{\mathrm{max},i}^{2}}{\alpha \, \mathcal{C}_{\mathrm{ys}}^{2}} = \frac{(\Delta \, \mathrm{K})_{,i}^{2}}{(1-\mathrm{R})_{\alpha}^{2} \, \mathcal{C}_{\mathrm{ys}}^{2}} ; \quad \mathbf{r}_{\mathrm{po}} = \frac{\mathbf{K}_{\mathrm{max},o}^{2}}{\alpha \, \mathcal{C}_{\mathrm{ys}}^{2}}$$
(3.3)

where K_{\max} is the maximum stress intensity in a given cycle, $\alpha = 2\pi$ for plane stress, and $\alpha = 6\pi$ for plane strain according to Irwin's plastic zone approach.

There is retardation as long as the current plastic zone size is contained within a previously generated plastic zone. If ai + $r_{pi} \ge ao + r_{po}$ the crack has grown through the overload plastic zone, and the retardation factor becomes $C_p = 1$ by definition. The power m in eq(3.2) has to be determined empirically. Wheeler finds m = 1.43 for D6ac steel and m = 3.4 for $T_i = 6A1 - 4V$.

For the case of a single overload in a constant amplitude test the retardation factor gradually decreases to unity while the crack progresses through the plastic enclave. If a second high load occurs, producing a plastic zone extending beyond the border of the existing plastic enclave, the boundary of this new plastic zone will have to be used in the equations (figure 3.2.b), and the instantaneous crack length will then become the new a₀.

The retarded crack-growth rate can be determined from the baseline (constant amplitude) crack growth rate as,

$$da/dN$$
) retarded = C_p(da/dN) linear (3.4)

where (da/dN) linear follows from constant amplitude data.

Predictions made by Wheeler by using his cycle-by-cycle integration method led to fairly good predictions of block-programme crack propagation tests. But the model does not account for compressive loading. Since the fracture surfaces are pressed together by compressive stresses, stress intensity has no physical meaning under compressive loading and therefore if the stress intensity factor is less than zero it is set at zero. On the other hand,although compressive peak hoad causes acceleration of crack growth rate, the Wheeler model does not account for this.

3.2. The Willenborg Model

The method was proposed by Willenborg, Engle and Wood (62). They also make use of the plastic enclave formed at the overload (figure 3.3). The plastic enclave extends to

$$a_{p} = a_{0} + r_{p0} = a_{0} + \frac{1}{\alpha} - \frac{K_{0}^{2}}{\zeta_{ys}^{2}}$$

where a_p is the distance from the centre of the crack to the boundary of the plastic enclave and the other symbols have the some meaning as in eqs(3.2) and (3.3). Willenborg et al. consider the stress intensity that would be required to produce a plastic zone (at the tip of the current erack a_i) that would extend to the border of the plastic enclave (figure 3.3).



Figure 3.3. The model of Willenborg, Engle and Wood (62)

This means that it has to be determined what magnitude of K_{max} is required to give:

$$a_{i} + r_{p.req} = a_{o} + r_{po}$$
 (3.6)

where $r_{p,req}$ is the plastic zone required to reach the boundary of the existing plastic enclave. The K_{max} req to achieve this, is given by:

$$\frac{1}{\alpha} \frac{\mathbf{x}_{\max.req}^2}{\mathbf{b}_{ys}^2} = \mathbf{a}_0 + \mathbf{r}_{po} - \mathbf{a}_i.$$
(3.7)

In the first cycle subsequent to the overload, a is still equal to

(3.5)

a. Hence K max.req would be equal to the stress intensity of the overload, as should be expected.

Willenborg et al. make the rather odd assumption that $K_{max.i}$ actually occuring at the current crack length a_i , will be effectively reduced by an amount K_{red} , given by:

$$K_{\text{red}} = K_{\text{max.req}} - K_{\text{max,i.}}$$
(3.8)

The residual compressive stresses introduced by an overload reduce the effective stress at the crack tip. This implies that the effective stress is the difference between the active stress and the residual stress. Eq(3.8) means that Willenborg et al. expect that the magnitude of the residual stress is given by:

$$G_{res} = \frac{K_{max.req}}{\sqrt{\pi a_i}} - \frac{K_{max.i}}{\sqrt{\pi a_i}}$$
(3.9)

This means that both $K_{max.i}$ and $K_{min.i}$ in cycle i are reduced by an amount $K_{red.}$ Hence, the effective stress intensity is given by:

K_{min.eff.i}= K_{min.i}- K_{red} = K_{min.i} + K_{max.i}- K_{max.req.} (3:10)

If either $K_{\text{min.eff}}$ or both $K_{\text{max.eff}}$ and $K_{\text{min.eff}}$ would be smaller than zero they are set at zero. If the latter accurs, $\Delta \dot{K}_{\text{eff.i}}$ will be smaller than ΔK_i : if not, $\Delta K_{\text{eff.i}} = \Delta K$, as can be seen from figure 3.3. The cycle ratio R_{eff} becomes

$$R_{eff} = \frac{K_{min.i} - K_{red}}{K_{max.i} - K_{red}} = \frac{K_{min.i} + K_{max.i} - K_{max.req}}{2K_{max.i} - K_{max.req}}$$
(3.11)

Both ΔK_{eff} and R_{eff} can be calculated and then da/dN can be calculated from the Forman equation based on effective values:

$$da/dN = \frac{C \Delta K_{eff}^{n}}{(1-R_{eff})K_{c} - \Delta K_{eff}}$$
(3.12)

Willenborg et al. also show integration results of block-programme crack propagation in good agreement test data. An objection against their model is that the assumption regarding the residual compressive stresses is doubtful.

3.3. The Crack Closure Model

Let us consider a growing fatigue crack (figure 3.4). During its growth the plastic zone is moving with the tip of the crack. It is also increasing in size and as a first approximation (central crack).

$$r_p \div K^2 \div a.$$
 (3.13)

That means that the plastic zone size is proportional to the crack length a. The same will be true for

reversed plastic deformation monotonic plastic deformation

Figure 3.4. Plastic deformation in front of and in the wake of crack

the reversed plastic zone. Since the monotonic plastic zone is considerably larger than the reversed plastic zone, the consequence

of the growing fatigue crack is that monotonic plastic deformation has been left in the wake of the crack. This deformation involves elongation in the loading direction. As a result of this elongation the crack will close (at least partly) during unloading, and after full unloading (P = o) compressive residual stresses will be present in the wake of the crack. It means that residual compressive stresses are transmitted through the crack, because the fracture surfaces are pressed together by the plastic deformation left in the wake of the crack.

The phenomenon that the upper and lower fracture surfaces of a fatigue crack come together before complete unloading (i.e. at p > o, tensile load) implies that the crack is no longer fully open. This phenomenon in the literature is referred to as "crack closure". It was first observed by Elber (21) and it is sometimes referred to as the Elber mechanism.

The most well known method to indicate crack closure is by COD (crack opening displacement) measurements. For crack opening between two points A and B, close to the edges of the crack and in the center of the panel, the relation for an infinite sheet is:

$$COD = 4\alpha G / E$$
(3.14)

It means that COD is linearly proportional to 6 (Hooke's law) and to the crack length a. For a finite sheet a geometry correction factor has to be added. Measurements on panels with fine saw cuts (Figure 3.5)



Figure 3.5. Crack closure measurement.

have confirmed the linear relationship with excellent agreement between measured slopes (G/COD) and theoretical values. If a similar test is carried out on a panel with a fatigue crack the G-COD record shows a non-linear part. Above point A the record is linear and the slope is in agreement with the crack length, which implies that the crack is fully open. Below point A the slope (tangent to \mathcal{O} - COD record) is larger, which implies that the panel behaves as a panel with a shorter crack. This indicates that the crack is partly closed. The stress corresponding to point Α is called the crack closure stress (\mathcal{L}_{cl}) or the crack opening stress (\mathcal{G}_{op}). A full loop of a COD record (figure 3.6) usually shows a slight hysteresis, but there is no doubt about the occurence of full crack opening at A' the onset of crack closure at A''. The existence of a non-linear part followed by a linear part is easily observed, but the problem is to determine accurately the point A where the transition occurs. Measurements suggest that



COD

Figure 3.6. A full loop of COD.

branch (A') gives a slightly better reproduction and a more unambiguous determination of the closure stress. Nevertheless it cannot be denied that it is difficult to achieve a high accuracy.

In order to increase the accuracy, Paris suggested a compensation method, illustrated by figure 3.7. Instead of recording the COD signal it is compensated by a signal that would have been obtained under full linear behavior (i.e. no crack closure). This leads to a vertical line as long as the crack is fully open. The compensated COD signal now allows a much larger application which will bring out the transition point A more clearly. If the amplification is selected too high, the linear part (vertical line) may become erratic.

Another possibility to improve the sensitivity of the crack closure measurement is to locate the COD meter more closely to the crack tip. The effect on the \mathcal{O} - COD record is shown in figure 3.8.





The non-linear part $(A^{''} - A')$ of the record is small and the transition is more easily observed. This method can be used for a particular crack length, but if the crack is growing the COD-meter has to be moved also. If the COD-meter is too close to the crack tip, plasticity effects may obscure the measurements.



Figure 3.8. The COD-meter is too close to the crack tip.

In 1970 W.Blazewicz carried out fatigue tests in Delf. He made ball impressions on 2024-T3 sheet specimens before the crack growth test was started (figure 3.9) (66). As a result there is a zone between the impressions with residual compressive stresses.



Figure 3.9. The Blazewicz's tests.

This caused a delay in the crack growth, but (surprisingly enough at that time) the delay was small during the growth through the zone between the impressions, whereas it was significant at a later stage. The explanation is that the deformations of the ball impressions were the cause of crack closure after the crack had grown through the affected zone.

Elber (21) suggested the following relation between crack closure and crack growth. During a stress cycle a fatigue crack will be partly or fully closed as long as $C < G_{Cl}$ (figure 3.10). He then suggested that the stress variation will contribute to crack extension only if

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which leads to the definition of an effective stress range:

$$\Delta C eff = C max - Ccl$$

(3.15)



Figure 3.10. The crack closure model.

and an effective stress intensity factor:

$$\Delta K_{eff} = C \Delta C_{eff} \sqrt{\pi a}$$
(3.16)

The crack rate was supposed to be dependent on Δ K_{eff.} only.

$$da/dN = f (\Delta K_{eff})$$
(3.17)

This relation includes the effect of the stress ratio R because crack closure (and thus ΔK_{eff}) will depend on R. For 2024-T3 material Elber found that **6** was approximately constant during a fatigue test, implying that **6** was independent of the crack length a. This is an empirical result. He defined the ratio:

$$U = \frac{\Delta K_{eff.}}{\Delta K} \left(= \frac{\Delta G_{eff.}}{\Delta G} \right)$$
(3.18)

and the test results indicated the relation:

U = 0.5 + 0.4 R

This is again an empirical result. Combining the above equations leads to

(3.19)

 $\log \Delta K_{eff.} = \log \Delta K + \log (0.5 + 0.4 R)$

Figure 3.11 includes a schematic of a simple spectrum in which the opening stress is seen to vary. The success



Figure 3.11. The variation of opening stress during variable amplitude cycling as described by Elber (21).

in using a closure model for the prediction of delay following an overload depends upon the assumptions made regarding the stabilization of closure load following the overload and the number of cycles of load required to accomplish "equilibrium" (67). For example, following one or a few overloads (figure 3.12) closure load of the subsequent series of cycles would be effected by the presence of the overload. If the overload would cause the opening load of the subsequent cycles to be increased, under these assumptions, crack growth could be completely arrested for some conditions. Assuming growth of a retarded nature did occur following the overload, some assumptions would have to be made as to the length of crack or zone over which the retardation would apply. As a first approximation, the yield zone produced by the overload might be tried.

(3.20)



Figure 3.12. Schematic of closure load variation following the application of a single overload.

Shih and Wei (68) have reported evidence to refute closure as the only phenomenon responsible for retardation.

In summary, closure appears to be a real phenomena, which can be rather readilyobserved, measured and quantified. It apparently is not the only phenomena which causes delay. Results from past studies have differed because of specimen type, thickness, precracking procedure, notch geometry and instrumentation.

3.4. The Root Mean Square (RMS) Approach

Barsom (69) attempted to determine the magnitude of constant -amplitude cyclic-load fluctuation that results in the same a versus N curve obtained under variable-amplitude cyclic-load fluctuation when both spectra are applied to identical specimens (including initial crack length). In other words, one of the objectives of his investigation was to find a single stress-intensity parameter, such as mean, modal, or root mean square, that can be used to define the crack-growth rate under both constant-and variable-amplitude loadings. The selected parameter must characterize the probability-density curve.

A good correlation between data obtained under constant amplitude and variable-amplitude random-sequence load spectra was obtained on the basis of the root mean square of the load distribution, where the root mean square is the square root of the mean of the squares of the individual load cycles in a spectrum. The combined crack-growth-rate data are presented in figure 3.13 as a function of $\Delta K_{\rm rms.}$ The data show that, within the limits of the experimental work, the average fatigue-crack-growth rates per cycle, da/dN, under variable-amplitude random-sequence stress spectra can be represented by

$$da/dN = C(\Delta K_{rms})^n$$

where C and n are constants and

$$\Delta K_{\rm rms} = \sqrt{\frac{\sum_{i=1}^{k} \Delta K_i^2}{n}}$$

The root-mean-square value of the stress-intensity factor under constant-amplitude cyclic-load fluctuation is equal to the stressintensity-factor fluctuation. Consequently, the average fatiguecrack-growth rate can be predicted from constant-amplitude data by using eq(3.21).

(3.21)

(3.22)





The root-mean-square stress-intensity factor, ΔK_{rms} , is characteristic of the load-distribution curve and is independent of the cyclic-load fluctuations (70).

Smith (71) and Swanson (72) have also obtained good correlations with similar spectrum characterizations for random loadings. In these studies of random-loading crack growth, stress spectra were all represented by a continuous, unimodal distribution, in particular, by a Rayleigh distribution function. (see appendix-C). The use of an RMS type of characterization to predict fatigue-crack growth should probabily be restricted to load histories which can be described by such distributions and in which sequence effects are not expected to be significant.

IV. FACTORS AFFECTING CRACK PROPAGATIONS

When predictions of crack propagation have to be made, data should be available relavent to the conditions prevailing in service. Such data may be hard to find. Fatigue crack propagation is affected by an endless number of parameters and the circumstances during the test will seldom be the same as in service. The influence of the environment is the most conspicuous. Tests are usually not performed under controlled environmental conditions and part of the scatter in fatigue data may be attributed to this fact.

4.1. Effect of specimen thickness :

Apart from the effects due to the transition from a 90° to a 45° plane, thickness has little effect on the rate of crack growth, provided that the crack front is sensibly straight through the thickness. Some tests (73) showed that, at short crack lengths, the crack growth rates tended to increase with sheet thickness, but were not affected at longer crack lengths.

The crack growth rate may increase as the maximum stress intensity factor in the fatigue cycle, K_{max} , approaches K_c , the fracture toughness of the material. K_c in general decreases as the thickness increa-

ses, reaching a minimum K_{lc}under plane strain conditions. This change in K_c with thickness can cause a corresponding thickness effect at high crack growth rates as illustrated in Fig.4.1





4.2 The effect of specimen orientation

(For the specimen orientation notation, see appendix E.) Figure 4.2 shows the rate of propagation to be somewhat higher for transverse (T-L) than for longitudinal. (L-T) specimens from 7075-T73510 extrusions; this was generally true for both plate and extrusions of all alloys and tempers. For the hand forgings, the rates were highest for specimens oriented in the short-transverse



Figure 4.2. Effect of specimen direction on fatigue-crack propagation rate for 7075-T73510 extruded panel (73). direction. These directional relationships were generally consistent with relative ratings with regard to fracture toughness (73). Data in figure 4.2 also illustrate the results of comparisons of specimens with machined and as-fabricated surfaces; no significant differences in rate of crack propagation were detected.

Figure 4.3 shows that fatigue crack in long - transverse specimens propagate substantially slower through the thickness (T-S) than in the longitudinal direction of the forging (T-L). Rates of propagation through the thickness were also shown to be lower for longitudinal (L-S)



Figure 4.3. Effect of orientation on rate of crack propagation of a 7079-T652 hand forging.

7079-T652 and long-transverse (T-S) 7075-T7352 specimens from the forgings than for the comparable flatwise directions (L-T and T-L, res pectively). In the short transverse direction (data not shown), the rate of propagation was found to be slightly lower for S-T specimens than for S-L specimens for both alloys. K_{1c} values were not determined for the L-S, T-S or S-T orientations so K_{1c} crack growth rate comparisons cannot be made. Davis et al. (74) did report lower values of K_{1c} for propagation in the L-S and T-S orientations than

in the L-T and T-L orientations for both 7075-T6 and 7079-T6 hand forgings, so for these orientations, the consistency in ratings may not exist.

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4.3. The effect of stress ratio and K_{max}

The effect of stress (or cycle) ratio, R, and the maximum stress intensity, K_{max} , was studied in chapter II. (pp.3-9).

4.4. Effect of temperature.

Temperature has considerable effects on material properties in general. It has a significant influence on fracture toughness also.However, it is impossible to isolate temperature effects from the effects of the manyy other parameters discussed. As an example consider the thickness effect. A plate of relatively low thickness may show plane stress behavior with inherent high toughness at room temperature. At low temperatures the material has a higher yield stress which causes the plastic zone to be smaller : then the plate may show transitional or even plane-strain behavior, with inherent lower toughness. Apart from the intrinsic effects of temperature on toughness, there is an indirect effect, due to the temperature dependence of yield strength.

The brittle-ductile transition of structural steels is well known from Charpy impact tests. A similar transition may be expected when considering toughness values. In view of the experimental problems of fracture toughness at temperatures different from ambient, attempts have been made (75,76) to estimate K_{1c} on the basis of the Charpy im pact-energy. It seems feasible that a correlation between Charpy energy and fracture toughness exists, since the former is also equivalent to a fracture energy through G_{1c} . However, this reasoning is doubtful since, the Charpy energy is the integrated energy for complete fracture of the specimen. On the other hand G_{rc} is the energy for the first infinitesimal crack growth.

Nevertheless correlations are found between toughness and Charpy energy, particulary in the area of low toughness. Figure 4.4 ser ves as an illustration. In fact



charpy impact energy (ft.lb)

Figure 4.4. Correlation between toughness and charpy impact energy (75).

the charpy test is a dynamic test and it may be more sensible to cor relate the impact energy with the dynamic fracture toughness. Although Charpy tests may be able to give an indication of the toughness variation, this type of test is basically not compatible with fracture mechanics principles. Therefore its applicability as a basis for decisi ons or conclusions concerning fracture behavior in the context of fracture mechanics is debatable.

Figure 4.5 shows that a steel indeed shows a transition from low to high toughness with increasing temperature. Alley steels and other materials usually show a gradual increase of toughness with temperature, followed by a decrease at temperatures approaching the melting point.



Figure 4.5. Temperature dependence of toughness of various steels (75,77)

4.5. Effect of loading frequency

Four specimens of 2024-T3 aluminum are tested each at a different frequency but with the same load ratio (78). The results show that the threshold decreases with increasing frequency(figure 4.6). This was an unexpected result since no effect of frequency had been previously noted below 200 Hz and it is the reverse of the trend that is expected from environmentally enhanced fatigue crack growth (79).

However, if the possibility of crack tip heating is considered one might expect such a shift. At the higher frequencies, heat may be generated by crack tip plasticity rapidly enough to produce a lecal rise in temperature. An increase in temperature might produce the lower threshold in 2024-T3 aluminum as it does for A533 steel(7). At lower frequencies heat conduction away from the crack tip would proceed rapidly enough to prevent any appreciable rise in the temperature near the crack tip.



Figure 4.6 Effect of frequency on fatique crack growth rates for 2024-T3 aluminum (78).

James, L.A(80)investigated the effect of frequency upon the fatique-crack growth of type 304 stainless steel at 1000 F. The results of this study are presented in figure 4.7 as a plot of da/dN versus ΔK . Several observations are immediately apparent :



5

3/2
 stress intensity factor range, ΔK, lb/in
 Figure 4.7. The effect of frequency on the crack growth behavior of solution-annealed type 304 stainless steel at 1000°F.

(1) There appears to be a portion of the curve where the fatigue-crack growth rate is independent of cyclic frequency;

(2) Within the region where the crack growth rate is frequency-dependent, decreasing the cyclic frequency has a profound effect in increasing the rate of crack propagation ;

(3) The scatter of data tends to increase somewhat with decreasing frequency ;

(4) in the region where the crack growth rate is dependent upon the frequency, the slopes of the curves representing the various frequency levels appear to be quite similar
(5) The latter observation suggests that the frequency de pendent behavior could be represented by a power law of the form

$$\frac{d\alpha}{dN} = A(f) [\Delta K]^{n}$$
(4.1)

where A(f) = some function of frequency, and n = constant for a given material/environment combination.

4.6. Effect of environment :

For the 7079-T652 hand forgings tested in a humidity of about 90 percent, the propagation rate is three times higher in the humid environment. For alloy 7075-T7352, the difference between the rates under the two environments is less even at the higher ΔK values(73). The greater sensitivity of 7079-T652 to moisture seems consistent with evaluations which show 7079-T6-type products to be susceptible to stress-corrosion cracking in normal atmospheres, where as 7075-T73-type products are not (81).

4.7. Effect of loading sequence

A fatigue cycle preceded by a load of higher magnitude produ ces less crack propagation than it does in the absence of the higher preload. This retardation phenomenon was taken into consideration in detail, in chapter III.

V. COMPUTER PROGRAM EVALUATING FATIGUE LIFE

5.1. Introduction

Because of weight and economy, crack growth retardation gains importance in variable amplitude fatigue. A computer program was developed to predict the crack growth propagation in specimens subjected to randomized block loading. The program also takes into account the retardation phenomenon. The Wheeler and Willenborg models were used for predictions. A cycle-by, cycle integration scheme was used. What the program does is to evaluate the number of cycles, flights or certain repeated stress history untill the critical crack length is reached. For the linear crack growth, Paris-Erdogan eq., Walker's formula and Forman's equations were set to use. Six different types of flaw were taken into consideration ; Center-cracked panel, single-edge notched specimen, double-edge crack, Bend-specimen, compact tension specimen(CTS) and surface flaw(see table 2.1,p.13). Detailed description will be given in Appendix-D.

5.2. Program Outline

The program consists of fifteen routines and a supervisory ro-

utine. Each subroutine will be discussed below. Much of the data transmitted internally in the program, AKYUREK-Iis through labeled COMMON blocks.

SUPERVISORY ROUTINE - Overall supervisory routine. This segment controls the calling, in order, all subroutines.

CADATA - Subroutine for the constant amplitude crack growth data and material properties.

SIMPSN - Constant amplitude crack growth integration scheme using simpson's composite integration formula (see Appendix-B)

DATA - Data input subroutine.

SIF - Evaluates the stress intensity factors for a given stress and crack length.

SFSIF - Subroutine for the surface flaw stress intensity evaluation.

PZS - Evaluates the plastic zone size for a given stress intensity and material.

ESIF - Subroutine which calculates the effective stress intensity factors for the Willenborg model.

GROWTH - Subroutine for linear crack growth.

WHEELR - Subroutine for the Wheeler retardation model.

WLLNBG - Subroutine for the Willenborg model.

OUTPUT - Prints out the crack growth information.

MAGFAC - Magnification factor evaluation for surface flaw (see Appendix-A).

DATFIT - Curve fitting subroutine using least squares method (see Appendix-A).

BLOCK - Subroutine for the block by block crack growth integration scheme.

BLOCK DATA - Initializes the labeled common blocks.



Figure 5.1. Simplified flow diagram for crack growth computation

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THRANT

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C ASSUME THAT THE FIRST CYCLE IS OVERLOAD CYCLE? AQ=AI ARPO=ARPI AKHAXO;AKHAX IF (NFT_MNE*6) GO TO 20 CO=CI CRPO=CRPI CKHAXO=CKHAX C SELECTION OF THE CRACK GROWTH INTEGRATION SCHEME C SELECTION OF THE CRACK GROWTH INTEGRATION SCHEME? C SELECTION OF CRACK GROWTH INTEGRATION SCHEME? C START THE APPLICATION OF THE LOADS AND C EVALUATION OF CURRENT CRACK LENGTHS? T066 C T067 T069 T070 T077 T077 T074 C T075 C T075 C T075 C T075 C Y079 C080 C080 C081 C082 C082 C085 C085 C1085 C1085 C1085 C1085 C1085 C1085 C1085 C1085 C1087 C1089 C10 JE IS THE TOTAL NUMBER OF FLIGTHS, GROUP OF BLOCKS OR NUMBER OF REPEATING SPECTRAT DO T IE=1, JE JH=0 JH=0 JA IS THE TOTAL NUMBER OF BLOCKS OR NUMBER OF CYCLES IN A FLIGTH OR SPECTRAT DO 2 HA=1,JA JH=JH+IS JB(JH) IS THE TOTAL NUMBER OF CYCLES IN THE JHTH BLOCKT KB=JB(JH)
 UB9

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 T092

 T093

 T094

 C

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 T003
 Ē. JB(JH) "IS THE TOTAL NUMBER OF CYCLES IN THE JH TH BLOCK" KB=JB(JH) DO 3 IB=1,KB NT IS THE TOTAL NUMBER OF CYCLES NTENT+1 STRESS INTENSITY EVALUATION IN AI DIRECTION? IF (NFT,NETG) CALL SIF STRESS INTENSITY EVALUATION FOR SURFACE FLAW IF (NFT,EQT6) CALL SFSIF PLASTIC ZONE SIZE EVALUATION FOR THE CURRENT CYCLE CALL PZS ++*CONTROL OF THE PLASTIC ENCLAVE FOR RETARDATION IN AI DIRECTION. IF (AD * ARPO_AIPARPI) 25,25,26 CURRENT CYCLE IS THE NEW OVERLOAD? AC = AI AKMAXO = AKMAX ARPO = ARPI IF (NFT,NETG) GO TO 30 +**CONTROL OF THE PLASTIC ENCLAVE FOR RETARDATION IN THE MAJOR AXIS DIRECTION OF THE SURFACE FLAW. CURRENT CYCLE IS THE NEW OVERLOAD IN THE MAJOR AXIS DIRECTION OF THE SURFACE FLAW. 1103 26 1110 26 1111 C CURRENT CYCEFTS THÉ NEW OVERLOAD IN THE HAJOR AXIS DIRECTION OF THE SURFACE FLAW. CO-CCI CO

SUBROUTINE FOR THE CONSTANT AMPLITUDE LOAD FATIGUE CRACK GROWTH DATA AND MATERIAL PROPERTIES LINEAR CRACK GROWTH DATA AND MATERIAL PROPERTIES C. IS THE CONSTANT IN THE LINEAR CRACK GROWTH EQUATIONS; C. NIS THE POWER CONSTANT IN THE LINEAR CRACK GROWTH EQUATIONS; C. SKC IS THE FRACTURE TOUGHNESS OF THE SPECIMEN; FEAD(NI,135), C;CN,SKC 135 FORMAT(3020714) C. SYS IS THE WIDTH OF THE SPECIMEN; C. MIS THE WIDTH OF THE SPECIMEN; C. MIS THE WIDTH OF THE SPECIMEN; C. MIS THE WIDTH OF THE SPECIMEN; C. MIS THE WIDTH OF THE SPECIMEN; C. MIS THE NUMBER OF FLAW TYPE; C. MIS THE NUMBER OF FLAW TYPE; C. SINGLE EDGE NOTCH; C. SINGLE EDGE CRACK; C. THE SINGLE EDGE SINGLE EDCE SINGLE EDCE SINGLE EDCE S 05 ŽŇŽ 209 1209 1210 **1**40 1211 1212 1213 1214 C

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 READINIDO LT

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 145
 FORMAT(20,17)

 1217
 WRITE(NO,150) WN

 1217
 WRITE(NO,150) WN

 1217
 FORMAT(22,42,3HWN=1020,14)

 1217
 S5

 1217
 FORMAT(22,42,3HWN=1020,14)

 1217
 FORMAT(22,42,3HWN=1020,14)

 1217
 S5

 1221
 C

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 C

 1222
 C

 MAJOR AXIS DIRCTION IN SURFACE FLAW

 1223
 READ(NI,145) CI

 1224
 165
 IF (NFT, NE 14) GO TO 170

 1224
 165
 IF (NFT, NE 14) GO TO 170

 1225
 READ(NI,145) CI
 READ SPECIMEN THE SUPPORTS IN BEND SPECIMEN

 1225
 READ(NI,145) Z
 READ(NI,145) Z

 1227
 170
 RETURN

 1228
 END
 STON OF THE TOTAL NUMBER OF CYCLES REQUI

 THE TOTAL NUMBER OF CYCLES REPUIRED TO PROPAGATE=

TURGUT

THE CRACK FROM INITIAL CRACK LENGTH, AI, TO CRITICAL OF FINAL CRACK LENGTH, AF, USING SIMPSON'S COMPOSITE INTEGRATION SCHEME CRACK LENGTH, AF, USING SIMPSON'S COMPOSITE INTEGRATION SCHEME SUBROUTINE SIMPSN DIMENSION F(9999) DOUBLE PRECISION ACP, ADAL, ADAR, ADK, AF, AI, AKMAX, AKMAXE, AKMAXO, AKMI EN, AKMINE, ALPHA, AO, AR, ARPI, ARPO, B, CICCP, CDAL, CDAR, CF, CI, CKMAX, CKMA EXO, CKMIN, CN, CO, CR, CRPI, CRPO, F, H, PHI, PI, QMAX, QMIN, SMAX, SMIN, RAB, RA ECISKC, SYSI, H, WM, WN, Z, BDAL, DDAL COMMON/TA2/ADAL, ADK, AF, AI, AKMAX, AKMAXO, AKMIN, ALPHA, AMF, AO, AR, ARPI E, ARPO, B, C, CDAL, CDK, CI, CKMAX, CKMAXO, CKMIN, CN, CO, CR, CRPI, CRPO, IA, IB E, IE, PJ, J, J, JE, KB, MN, MP, HR, NC, NFT, NI, NO, NP, PI, SKC, SYS, W, NH, WN, Z, A EDAR, CDAR, NT H IS THE STEP SIZE FOR NUMERIC INTEGRATION H=0,1D-0L DATA SUBH, TOT/2*0,0/ 2*N NUMBER OF STEPS N=(AFFAI)/(2*H) IR=2*N+T GOAL SIF CALC SIF CALC SIF CAC SUBROUTINE SIMPSN 1250 1251 C 1252 1253 1254 1255 1255 1255 1257 200 1258 1259 1260 21 200 CONTINUE IS=2*N DO 22 JR=2,IS,2 SUM=SUM+F(JR) CONTINUE SUM=4**SUM+F(I)+F(IR) IT=2*N-T DO 23 TU=3,IT,2 TOT=TOT+F(IJ) CONTINUE TOT=2*TOT NTOTAL=(SUM+TOT)*H/3 WRITE(NO,2D5) NTOTAL FORMAT(//,2X,2IHLIFE OF THE SPECIMEN=,I20,2X,6HCYCLES) RETURN END 1261 1262 1263 264 22 1515678911567891 77777778911 222222278882345678 2222222288845678 2222222288845678 EDATA INPUT SUBROUTINE SUBROUTINE DATA DOUBLE PRECISION ACP, ADAL, ADAR, ADK, AF, AI, AKMAX, AKMAXE, AKMAXO, AKMI LN, AKMINE, ALPHA, AO, AR, ARPI, ARPO, B, C, CCP, CDAL, CDAR, CF, CI, CKMAX, CKMA Exo, CKMIN, CN, CO, CR, CRPI, CRPO, F, H, PHI, PI, OMAX, QMIN, SHAX, SMIN, RAB, RA CC, SYS, N, WH, WH, Z, BDAL, DDAL COMMON/TA1/A(8), JBCAL, DDAL COMMON/TA2/ADAL, ADK, AF, AI, AKMAX, AKMAXO, AKMIN, ALPHA, AMF, AO, AR, ARPI COMMON/TA2/ADAL, ADK, AF, AI, AKMAX, AKMAXO, AKMIN, ALPHA, AMF, AO, AR, ARPI CARPO, B, C, COLL, COK, CI, CKMAX, CKMAXO, CKMIN, CN, CO, CR, CRPI, CRPO, IA, TB CARPO, B, C, COLL, COK, CI, CKMAX, CKMAXO, CKMIN, CN, CO, CR, CRPI, CRPO, IA, TB CARPO, B, C, COLL, COK, CI, CKMAX, CKMAXO, CKMIN, CN, CO, CR, CRPI, CRPO, IA, TB CARPO, B, C, COSTANT USED IN EVALUATING THE PLASTIC ZONE SIZE_ ALPHA IS, A CONSTANT USED IN EVALUATING THE PLASTIC ZONE SIZE_ ALPHA IS, A CONSTANT USED IN EVALUATING THE PLASTIC ZONE SIZE_ ALPHA IS, A CONSTANT USED IN EVALUATION MODEL NUMBER[®] HP IS PLASTC ZONE SIZE EVALUATION MODEL NUMBER[®] HP IS DLASTC ZONE SIZE EVALUATION MODEL NUMBER[®] HP IS DISDALE MODEL_ TZ IRWIN MODEL HODEL STREAMS CONSTANT USED IN EXPONENT. READIN: 300) ALPHA, HP, JE, JA FORMAT(V, IX, GALPHA, HP, JE, JA FORMAT(V, IX, GALPHA, HP, JE, JA FORMAT(V, IX, GALPHA, HP, JE, JA FORMAT(V, IX, GALPHA, HP, JE, JA FORMAT(V, IX, SHALPHA, DZO, 14, 9X, 3HMP=, IIG, 13X, 3HMR=, IID) IF (HR®NE, I' GO TO 311 WH IS THE WHEELER RETARDATION EXPONENT. READIN; 310) WH FORMAT(V, IX, SHAME, DZO, 14, 9X, 3HMI=, IGO) ***STRESS OR LOAD HISTORY INPUT[®] WR ITE(NO, 355) FORMAT(IH, IZ, 2LIB, IOX, 4HSMAX, 18X, 4HSMIN) WR TE(NO, 355) FORMAT(IH, IX, 2X, 2LIB, IOX, 4HSMAX, 18X, 4HSMIN) WR TE(NO, 350) FORMAT(IH, IX, 2Z, 2LIB, IOX, 4(1H=), 18X, 4(1H=)) DO 31 LA=1, JA CCCCCC 30 300 301 302 303 304 305 С 305 305 307 309 310 30, 1309 3. 1309 201 1310 311 1310 315 1312 1312 1315 1315 1315 1315 1315 1315 1315 1315

TURGUT

 315 C
 JB(LA) IS THE NUMBER OF CYCLES IN THE LATH BLOCK?

 336 C
 SHAX(LA) IS THE MAXIMUM STRESS OR LOAD IN THE LATH BL

 137 C
 SHIN(LA) IS THE MAXIMUM STRESS OR LOAD IN THE LATH BL

 137 C
 SHIN(LA) IS THE MAXIMUM STRESS OR LOAD IN THE LATH BL

 137 C
 SHIN(LA) IS THE MINIHUM STRESS OR LOAD IN THE LATH BL

 137 C
 SHIN(LA) IS THE MINIHUM STRESS OR LOAD IN THE LATH BL

 137 C
 SHIN(LA) IS THE MINIHUM STRESS OR LOAD IN THE LATH BL

 137 S
 325
 FORMAT(110, 2020-14)

 1320
 WRITE(NO, 330) JBLA), SHAX(LA), SHIN(LA)

 1321 ST
 CONTINUE

 1322 ST
 CONTINUE

 1323 ST
 CONTINUE

 1324 335
 FORMAT(1/, 3X, 3HJE=, ILD, 2X, 3HJA=, IS)

 1325 ST
 HRITE(NO, 335) JE, JA

 1326 ST
 FORMAT(1/, 3X, 3HJE=, ILD, 2X, 3HJA=, IS)

 1327 C
 IP ST PRINTS OUT INFORMATION AT THE END OF EACH BLOCK,

 1329 C
 IF (HI E2, 2) GO TO 337

 1329 C
 IP ST PRINTS OUT INFORMATION AT THE END OF EACH FLIGTA

 1330 C
 IP ST PRINTS OUT AT THE END OF EVERY NP CYCLES NP IS

 1331 C
 GIVEN AS INPUT

 1332 C
 IVEN AS INPUT

 1333 340 FORMAT(I10)
 IP BLOCK

THRAUT

COMPACT TENSION SPECIMEN(CTS) PSMAX AND SMIN ARE THE LOADS P(NFT=5 AM MAXES MAX(IA)/IB*SSGT(M1)) CT P(A)/IS*CSGT(A)/IS*CSGT(A)/IS*CSGT(A)/IS*CT_5)+65577*((AI/W)**2_5) Pd(0) CT (A)/IS*CSGT(A)/IS*CSGT(A)/IS*CST(A)/IS*CT_5)+65577*((AI/W)**2_5) Pd(0) CT (A)/IS*CSTANA (A)/IS*CSGT(A)/IS*CSTANA (A)/IS*CT_5)+65577*((AI/W)**2_5) Pd(0) CT (A)/IS*CSTANA (A)/IS*CSGT(A)/IS*CSTANA (A)/IS*CT_5)+65577*((AI/W)**2_5) Pd(0) CT (A)/IS*CSTANA (A)/IS*CSGT(A)/IS*CSTANA (A)/IS*CSTANA	CR=CKHIN7CKMAX GO TO 5D5 DC CP=0,0000 D5 RETURN END ==================================	AVULDING FRODUC IF (CKMAX LTOCO) C STRESS INTENSITY R CDK=CKMAX CKMIN CYCLE RATIO IN THE IF (CKMAX EQ.00) G	AUK=AAA CYCLE R IF(AKMA AR=AKMI GO TO 4 AR= 0,0 0						2	
COMPACT TENSION SPECIMEN(CTS) FSMAX AND SHIN ARE THE LOADS F(NFT=5 AM MAXES MAXIAA/IB*ESGRIUN) AND AND AND AND AND AND AND SHIP ARE THE LOADS F(NFT=5) AM MAXES MAXIAA/IB*ESGRIUN) ATTACT AND AND AND AND AND AND AND AND AND AND	CR=CKHIN7CKMAX GO TO 505 CP=O,0D0D RETURN END =SUBROUTINE WHI =SUBROUTINE WHI =SUBROUTINE WHI =SUBROUTINE WHI	IF (CKMIN, LT OC) C IF (CKMIN, LT OC) C STRESS INTENSITY R CDK=CKMAX, CKMIN CYCLE RATIO IN THE IF (CKMAX, E9, 00) G	ADK=AKH CYCLE R IF(AKMA AR=AKMI GO TO 4 AR= 0,00					26	0 5	
COMPACT TENSION SPECIMEN(CTS) #SMAX AND SHIN ARE THE LOADS #(NFT=5 AMMAXESTAX([A]/(B+DSGRT([J])) AND ALL STAX([A]/(B+DSGRT([J])) AND ALL STAX([A]/(B+DSGRT([J])) AND ALL STAX([A]/(B+DSGRT([J])) AND ALL STAX([A]/(B+DSGRT([J])) FT (A ALL ALL STATES THE COMPACT ALL STATES AND SHIN ARE THE LOADS #((AI/W)++2_5)#GL AMMAXEC #AKMAX AND ALL STATES AND COMPRESSIVE LOADING. FT (A ALL ALL STATES AND ALL AND ALL STATES AND SHIN ARE THE LOADING. FT (A ALL ALL STATES AND ALL AND ALL STATES AND ALL STATE	CR=CKMIN7CKMAX GO TO 505 CR=0,0000 RETURN END ==================================	IF (CKMAX LT 00) C IF (CKMAX LT 000) C STRESS INTENSITY R CDK=CKMAX CKMIN CYCLE RATIO IN THE IF (CKMAX E9:00) G	ADK=AKM CYCLE R IF(AKMA AR=AKMI GO TO 4 AR= 0,00						1	
DPACT TENSION SPECIHEN(CTS) PSMAX AND SMIN ARE THE LOADS=(NFT=5 (MAX=SMAX(IA)/(B*DSORT(W)) MTMESTIN(IA)/(B*DSORT(W)) MTMES	R=CKMIN7CKMAX D TO 505 FTO 7000 ETURN ND SUBROUTINE WHI SIZE FOR A GIV	F(CKMAXLTTOTO) F(CKMAXLTTOTO) TRESS INTENSITY R DK=CKMAXTCKMIN YCLE RATIO IN THE F(CKMAXLED)OOD G	DK=AKH YCLE R F(AKMA R=AKMI 0 T0 4 R= 0.00	SAASCICALLS	LE NE GOEC	SDNXCCC	H H H	III SACIAGARE	A1 C(C) A1	C
PACT TENSION SPECIHEN(CTS)=SMAX AND SHIN ARE THE LOADS=(NFT=5 AX=SMAX(1A)/(B=DSQRT(M)) TYPESTON CONFERENCE (I)) TYPESTON	CKMIN7CKMAX TO 505 0,0000 URN UBROUTINE WHI IZE FOR A GIV	CKMINS FK95 CO) C CKMINS LT OCO) C ESS INTENSITY R ECKMAX CKMIN LE RATIO IN THE CKMAX EQ OCO G				UBUA -SMMREP	S U	* C C RKC C = TD		он
ACT TENSION SPECIMEN(CTS) FSMAX AND SHIN ARE THE LOADS=(NFT=5) CTS MAX(IA)/IB+DSQRT(W)) TS MAX(IA)/IB+DSQRT(W)) TS MAX(IA)/IB+DSQRT(W)) TS MAX(IA)/IB+DSQRT(W)) TS MAX(IA)/IB+DSQRT(W)) TS MAX(IA)/IB+DSQRT(W)) TS MAX(IA)/IB+DSQRT(W)) TS MAX(IA)/IB+DSQRT(W)) TS MAX(IA)/IB+DSQRT(W)) TS MAX(IA)/IB+DSQRT(W)) TS MAX(IA)/IB+DSQLTWATCOM TS	KMIN7CKMAX D 505 d000 TN BROUTINE WHI ZE FOR A GIV	KHAN LT OTO CO KHAX LT OTO CO SS INTENSITY R CKMAX CKMIN E RATIO IN THE KHAX LEQ OCO CO G	AKH E R KMA KMI 0 4	EATEATTAAE	IIICWXNNL	RBKCKMMP	5		ATR2 (AT	P
T TENSION SPECIHEN(CTS) PSMAX AND SHIN ARE THE LOADS P(NFT=5 SMAX(IA)/(B+DSORT(H)) SMINITAL/(B+DSORT(H)) SMINIT	INFCKMAX 505 5000 FOR A GIV	IN LT OLO CONCLUSION CONCLUSICONCLUSICONCLUSICONCLUSICONCLUSICONCLUSICONCLUSICONCLUS CONCLUSICONCLUS CONCLUSICONCLUS CONCLUSICONCLUS CONCLUSICONCLUS CONCLUSICONCLUS CONCLUSICONCLUS CONCLUS					RO	VANSAEK(D.N		
TENSION SPECIMEN(CTS)FSMAX AND SMIN ARE THE LOADS=(NFT=5 MAX(IA)/IB+DSORT(W)) MIN(IA)/IB+DSORT(W)) MIN(IA)/IB+DSORT(W)) MIN(IA)/IB+DSORT(W)) MIN(IA)/IB+DSORT(W)) MIN(IA)/IB+DSORT(W) MIN(NJCKMAX D5 OD JTINE WHI FOR A GIV	ATIO IN THE X EQ QOD G			II*HHHIA	IT NIS//B		IIA MRAI4D	5516/00	T
TENSION SPECIMEN(CTS) FSMAX AND SHIN ARE THE LOADS F(NFT=5 AX(IA)/(B+DSORT(W)) NI(AI)/(B+DSORT(W)) NI(AI)/(B+DSORT(W)) FACTOR EVALUATION FOR CTS (AI)/(B+DSORT(W)) FACTOR EVALUATION FOR CTS (AI)/(B+DSORT(W)) FACTOR SUBJECTS((AI)/W) FACTOR SUBJECTS((AI)/W) FACTOR SUBJECTS(CAI)/(AI)/(AI)/(AI)/(AI)/(AI)/(AI)/(AI)/(CKMAX D Tine whi or a giv	LT:0:0) C LT:0:0) C RTENSITY R X=CKMIN T10 IN THE LEQ:0:0) G	AAX NS D	L'HI SNXI	C/ ZALICO	IRENYTT JA	Ī			•
ENSION SPECIHEN(CTS) FSMAX AND SHIN ARE THE LOADS F(NFT=5 (IA)/(B+DSORI(U)) (IA)/(B+DSORIUE) (IA)/(CKMAX IIIIII INI WHI IIIIIIIII IIIIIIIII IIIIIIIII	TTOTO) C TTOTO) C TENSITY R CKMIN CO IN THE EQ.Q.O.O) G	×T .750	NITNET CON	7 2)P**AF	NIR I SAAC	=:	I ANXT POD	A 10() ##	TE
SION SPECIHEN(CTS) #SMAX AND SMIN ARE THE LOADS #(NFT=5 IA)/(B*DSORT(U)) FACTOR EVALUATION FOR CTS T/giveDSORT(IA)) FACTOR EVALUATION FOR CTS T/giveDSORT(IA)) FACTOR EVALUATION FOR CTS T/giveDSORT(IA)) FACTOR COMPRESSIVE LOADING FACTOR COMPRESSIVE LOADING FACTOR COMPRESSIVE LOADING FACTOR AT DIRECTION. FACTOR AT A COMPANY AND AND AND AND AND AND AND AND AND AND	MAX HAX HI A GIV	IN THE					NE			N
ION SPECIMEN(CTS) FSMAX AND SHIN ARE THE LOADS F(NFT=5 A) (B*DSQRI(M)) A) (B*DSQRI(M)) A) (D*DSQRI(M)) A)	AX ===== WH1 GIV =====	0000) 0000) SITY R HIN IN THE 0000 0		NAA NSSO IN	T+20110	SID	-	RONK ION	LIFHSMM	IS
ON SPECIHEN(CTS) #SMAX AND SHIN ARE THE LOADS #(NFT=5) //B+DSORT(W)) //B+DSORT(W)) //B+DSORT(A)) //B+DSORT(X WHI GIV ===	ITY R N THE		SHHSHHMOOS	E() ADON	FSHCW(DAJ	F F	FOOSHFOA	AAA/ TAI	I
<pre>N SPECIHEN(CTS)#SMAX AND SHIN ARE THE LOADS=(NFT=5 /(B*DSORT(W)) /(B*DSORT(W)) /(B*DSORT(W)) /(B*DSORT(W)) /(B*DSORT(W)) /(B*DSORT(W)) /*(C)/PRESSIVE LOADING_ /*(C)/PRESSIVE LOADING_////////////////////////////////////</pre>		D) C D) C TY R THE D) G	N X	IFFIAI SOIT	R	SIAO M8ALE	= : 0 = :	RAIIO))CW5XN	01
SPECIMEN(CTS) =SMAX AND SHIN ARE THE LOADS =(NFT=5 B*DSORT(U)) PEOPLUATION FOR CTS 				T**TXNC001)	R 1 AP127		R		/(/() //	N.
PECIHEN(CTS) #SHAX AND SHIN ARE THE LOADS #(NFT=5 #DSORT(4)) #DSORT(4)) #DSORT(4)) #DSORT(4)) #DSORT(4)) #CVALUATION FOR CIS #SSETUE LOADING_ AKHAN=0,0000 AKHAN=		CCR EG	Ţ⊦)'	757 57 (1) (1) (1)			S	1) / A		s
ECIMEN(CTS) SHAX AND SHIN ARE THE LOADS (NFT=5) DSORT(4)) STATUSON FOR CIS 50 E105554 (11/4) + T ₁ 5) + 65577 + ((AI/M) + +2,5) = 40; 60 + 7(AI/M) + +445 DMPRESSIVE LOADINS: KHAN=0,0000 KHAN=0,0000 KHAN=0,0000 ANGE FOR AI DIRECTION; DIRECTION: 0 TO 426 CP, ADAL, ADAR, ADK, AF, AT, AKMAX, AKMAXE, AKMAXO, AKM CP, ARPTIARPO, B, CCCP, CDAL, CDAR, CF, CI, CKMAX, CKM CP, CACO, F, H, H, PI, DAL, OHA, SHAX, SHAX, SHAX, SHAX, RA 7, BDAL, DDAR, ADK, AF, AT, AKMAX, AKMAXE, AKMAXO, AKM CP, ARPTIARPO, B, CCCP, CDAL, CDAR, CF, CI, CKMAX, CKM 7, BDAL, DDAR, ADK, AF, AT, AKMAX, AKMAXE, AKMAXO, AKM 7, BDAL, ADAR, ADK, AF, AT, AKMAX, AKMAXE, AKMAXO, AKM 7, BDAL, ADAR, ADK, AF, AT, AKMAX, AKMAXE, AKMAXO, AKM 7, BDAL, ADAR, ADK, AF, AT, AKMAX, AKMAXE, SHAX, SHAX, SHAX, SHAX, RA 7, BDAL, ADAR, ADK, AF, AT, AKMAX, AKMAXE, AKMAXO, AKM 7, BDAL, ADAR, ADK, AF, AT, AKMAX, AKMAXE, AKMAXO, AKM 7, BDAL, ADAR, ADK, AF, AT, AKMAX, AKMAXE, AKMAXO, AKM 7, BDAL, ADAR, ADK, AF, AT, AKMAX, AKMAXE, AKMAXO, AKM 7, BDAL, ADAR, ADK, AF, AT, AKMAX, AKMAXE, AKMAXO, AKM 7, BDAL, ADAR, ADK, AF, AT, AKMAX, AKMAXE, AKMAXO, AKM 7, BDAL, ADAR, ADK, AF, AT, AKMAX, AKMAXE, AKMAXO, AKM 7, BDAL, ADAR, ACK, AKMAX, AKMAX, AKMAXE, AKMAXO, AKM 7, BDAL, ADAR, AKA, AKMAXO, AKMAX, AKMAXE, AKMAXE, AKMAXO, AKM 7, BDAL, ADAR, AKA, AKMAXO, AKMAX, AKMAXE, AKMAXE, AKMAXO, AKM 7, BDAL, ADAR, AKIS, AKMAXO, AKMAX, AKMAXE, AKMAX	I H N I		G			A , , , BDK ,	U	CAA R IG	*	P
CIHEN(CTS) PSMAX AND SHIN ARE THE LOADS P(NFT=5 SGRT(H)) SGRT(H)) SGRT(H)) SGRT(H)) SGRT(H)) SGRT(H)) PADAT TON FOR CIS PHEAT TON FOR CIS PHEAT TON FOR CIS PHEAT TON FOR THE STATE STATE (AI/W) ++2 s) PHEAT HAX=0 2000 HAX=0 426 P, ADAL, ADAR, ADK, AF, AI, AKMAX, AKMAXE, AKMAXO, AKM RP1, CR60, F, H, PHI, PI; SHAX; GHIN; SHAX; SHIN, RAB, R P00), R(B), SMAX(90U), SMIN(90D) AF, AI, AI, AKMAX, AKMAXO, AKMIN, ALPHA, AMF, AO, AR, ARP P00), R(B), SMAX(90U), SMIN(90D) AF, AI, AKMAX, AKMAXO, AKMIN, ALPHA, AMF, AO, AR, ARP P00), R(B), SMAX(90U), SMIN(90D) St(ICI++2=AI++2)/CT++2)=(37/647)+((CI++2=AI++ P EVALUATION SKA(IA)/SYS)++2 SMAX(IA)/SYS)+2 SMAX(IA)/SYS)+2 SMAX(IA)/SYS)+2 SMAX(IA)/SYS)+2 SMAX(IA)/SYS)+2 SMAX(IA)/SYS)+2 SMAX(IA)/SYS)+2 SMAX(IA)/SYS)+2 SMAX(IA)/SYS)+2 SMAX(IA)/SYS)+2 SMAX(IA)/SYS)+2 SMAX(IA)/SYS)+2 SMAX(IA)/SYS)+2 SMAX(IA)/SYS)+2 SM	=		0	VXNVJJRKKA	02 E((0	CACZ(K)M	= R	OKKA O		E
EHEN(CTS) SMAX AND SHIN ARE THE LOADS (NFT=5) SRI(H)) ALUATION FOR CTS LATION FOR CTS LATION FOR CTS LATION FOR CTS STEPSE (LAT/W)**1,5)*05557*((AI/W)**2,5)=40, PRESSIVE LOADING. STEPSE (LAT/W)**1,5)*05557*((AI/W)**2,5)=40, STEPSE (LAT/W)**1,5)*05557*((AI/W)**2,5)=40, PRESSIVE LOADING. STEPSE (LAT/W)**1,5)*05557*((AI/W)**2,5)=40, STEPSE (LAT/W)**1,5)*05557*((AI/W)**2,5)=40, STEPSE (LATION: ADAL, ADAR, ADK, AF, AI, AKMAX, AKMAXE, AKMAXO, AKM ARPE, ARPO, B, C: CC, CC, CDAL, CDAR, F, C: C: CKMAX, CKM ATRPI, ARPO, B, C: CC, CC, CDAL, CDAR, F, AX, S, WIN, FAB, B, B, D, C, CKMAX, CKM, CKM, C, C, CR, S, S, S, W, W, WN, 2, C, CKMAX, CKMAX, S, KHIN, ALPHA, AMF, AO, AR, ARP ADAL, DDAL, DDAL, D, SMIN(900) AF, AI, AKMAX, AKMAXO, AKMIN, ALPHA, AMF, AO, AR, ARP C, CKMAX, CKMAXO, CKMIN, C, C, C, CR, CRP T; CFPO, IA, T MP, MR, NC, NFT, NI, NO, NP, PI, SXC; SYS, W, WH, WN, 2, C THE SECOND KIND +(ICI*+2=AI+*2)/CI*+2)=(137/547)*((CI*+2=AI**)) EVALUATION AX(IA)/SYS)**2 (AMF) EVALUATION: ATION FOR MINOR AXIS(AI) DIRECTION: ATOON FOR MINOR AXIS(AI) DIRECTION: SSIVE LOADING IN AXIS(I) DIRECTION: AX=0,0000 SSIVE LOADING IN CI DIRECTION: AX=0,0000 SSIVE LOADING IN CI DIRECTION: AXOR AXIS DIRECTION: ALOULATES THE PLASTIC ZONE IN THE MINOR AXIS DIRECTION: AAUOR AXIS DIRECTION: ALOULATES THE PLASTIC ZONE ALOULATES	=: C S =:	M M N M	M;	A((APPEN HN	F 5 P SI SI R		= : F /	MF M N D	SCV5	C]
NEN(CTS) PSMAX AND SHIN ARE THE LOADS P(NFT=5 NT(M)) UATION FOR CTS UATION FOR CTS UATION FOR CTS DESSIVE LOADING. PORODO FOR AI DIRECTION? RECTION: 1 426 NDAL, ADAR, ADK, AF, AI, AKHAX, AKMAXE, AKHAXO, AKH NRPIARPOBEC CCP, COAL, COAR, CF, CI, CKMAXCAN, AKH NRPIARPOBEC CCP, COAL, COAR, CF, CI, CKMAXCAN, AKH NRPIARPOBEC CCP, COAL, COAR, CF, CI, CKMAXCAN, AKH NRPIARPOBEC CCP, COAL, COAR, CF, CI, CKMAXCAN, AKH NAL, DOAL 1 426 NDAL, ADAR, ADK, AF, AI, AKHAX, AKMAXE, AKHAXO, AKH NRPIARPOBEC CCP, COAL, COAR, CF, CI, CKMAXCAN, AKH NRPIARPOBEC CCP, COAL, COAR, CF, CI, CKMAXCAN, AKH NRPIARPOBEC CCP, COAL, COAR, CF, CI, CKMAXO, AKH NRPIARPOBEC CCP, COAL, COAR, CF, CI, CKMAXCAN, AKH NRPIARPOBEC CCP, COAL, COAR, CI, STAN, SHAN, AB, R D, CROO, KINO, CO, CR, CF, CF, TCRPOIA, AT NAL, NC, NFT, NI, NO, NP, PI, SKC, SYS, M, WH, WA, Z, THE SECONO KINO (CIAYSYS) **2 NHF) EVALUATION X(IA)/SYS) **2 NHF) EVALUATION X(IA)/SYS) **2 NHF) FOR MAJOR AXIS(AI) DIRECTION. ATION FOR MINOR AXIS(AI) DIRECTION. ATION FOR MANDA AXIS DIRECTION. STVE LOADING IN CI DIRECTION. HOR AXIS DIRECTION. SIVE LOADING IN CI DIRECTION. SIVE LOADING IN CI DIRECTION. SIVE LOADING IN CI DIRECTION. SIVE LOADING IN CI DIRECTION. SIVE LOADING IN CI DIRECTION. SIVE LOADING IN CI DIRECTION. SIVE LOADING IN CI DIRECTION. SIVE LOADING IN CI DIRECTION. SIVE LOADING IN CI DIRECTION. SIVE LOADING IN CI DIRECTION. SIVE LOADING IN CI DIRECTION. SIVE LOADING IN CI DIRECTION. SIVE LOADING IN CI DIRECTION. SIVE LOADING IN CI DIRECTION. SIVE LOADING AXIS DIRECTION. SIVE LOADING IN CI DIRECTION. SIVE LOADING AXIS DIRECTION. SIVE LOADING AXIS DIRECTION. SIVE LOADING IN CI DIRECTION. SIVE LOADING IN CI DIRECTION. SIVE LOADING IN CI DIRECTION. SIVE LOADING AXIS DIRECTION. SIVE LOADING AXIS DIRECTION. SIVE LOADING AXIS DIRECTION. SIVE LOADING AXIS DIRECTION. SIVE LOADING AXIS DIRECTION. SIVE LOADING AXIS DIRECTION. SIVE LOADING AXIS DIRECTION. SIVE LOADING AXIS DIRECTION. SIVE SING SIVENSITY AND HATERIAL = SIVENSITY AN		SINASE AL			+ (H) H) (/	BCCAF	AC	SE IFC		E۲
N(CTS) SMAX AND SHIN ARE THE LOADS (NFT=5 (M)) ATTON FOP CIS BSSS*(IAT/W)**ft5)+65557*((AI/W)**2,5) 40,0 (AI/W)**44,5) SSIVE LOADING. 000000 FOR AI DIRECTION? CTION. 426 AL, ADAR, ADK, AF, AI, AKMAX, AKMAXE, AKMAXO, AKM PTARPO, BCC, CC, CDAL, CDAR, CF, CI, CKMAA, CKM CROD, F, M, PHI, FI, SMAX, OMIN, SHAX, SMIN, RAB, R L, DDAL R(B), SMAX(90U, SMIN(90D) I, AKMAX, AKMAXO, AKMIN, ALCHA, AMF, AO, AR, ARP KMAX, CKMAXO, CKMIN, CN, CO, CR, CRP 1, CPO, IA, I R(B), SMAX(90U, SMIN(90D) I, AKMAX, KMMAXO, AKMIN, ALCHA, AMF, AO, AR, ARP KMAX, CKMAXO, CKMIN, CN, CO, CR, CRP 1, CPO, IA, I R(B), SMAX(90U, SMIN(90D) I, AKMAX, KMMAXO, AKMIN, ALCHA, AMF, AO, AR, ARP KMAX, CKMAXO, CKMIN, CN, CO, CR, CRP 1, CPO, IA, I R(B), SMAX(90U, SMIN(90D) I, AKMAX, AKMAXO, AKMIN, ALCHA, AMF, AO, AR, ARP KMAX, CKMAXO, CKMIN, CN, CO, CR, CRP 1, CPO, IA, I R(AI/SYS), **2 (IAJ/SYS), **2 (IAJ/SYS), **2 (IAJ/SYS), **2 (IAJ/SYS), **2 ISOART(PI*AI/OMAX) TION FOR MINOR AXIS(AI) DIRECTION. R(PI*(AI**2/CI/GMAX) TION FOR MINOR AXIS (CI) DIRECTION. R(PI*(AI**2/CI/GMAX) R(PI*(AI**2/CI/GMAX) VE LOADING IN CI DIRECTION. R AXIS DIRECTION. R AX			10				E			IE.
(CTS) PSMAX AND SHIN ARE THE LOADS P(NFT=5 M) TION FOR CIS TION FOR CIS AIJW)**4453 SIVE LOADINS. \$0000 OR AI DIRECTION? TION. 26 LADAR, ADK, AF, AI, AKMAX, AKMAXE, AKMAXO, AKM HARPO, BACICC; DAL, CDAR, CF CT, CKMAXC, KMAXO, AKM ROD, F, H, HI, PI, DHAX, OHIN, SHAX, SMIN, RAB, R ROD, SMAX(90U), SHIN(90D) IAXMAX, AKMAXO, IAXMIN, ALPHA, AMF, AO, AR, ARP MAX.CKMAXO, CKMIN, CN, CO, CR, COPT, CRPO, IA, I IAXMAX, AKMAXO, XKMIN, ALPHA, AMF, AO, AR, ARP MAX.CKMAXO, CKMIN, CN, CO, CR, COPT, CRPO, IA, I IAXMAX, AMAXO, CKMIN, CN, CO, CR, COPT, CRPO, IA, I IAY, NC, NT, FI, NO, NO, NP, PISKC, SNS, N, NM, WN, 2, SECOND KIND I**2_AI**2)/CT**2) =(37/647)*((CI**2#AI** LUATION ION FOR MINOR AXIS(AI) DIRECTION. DSGRT(PI*AI/OMAX) ION FOR MINOR AXIS(CI) DIRECTION. 100 FOR MINOR AXIS DIRECTION. 40000 M THE MINOR AXIS DIRECTION. AXIS DIRECTION. AXIS DIRECTION. AXIS DIRECTION. AXIS DIRECTION. AXIS DIRECTION. AXIS DIRECTION. AXIS DIRECTION. 40000 M THE MINOR AXIS DIRECTION. AXIS DIRECTION. 40000 M THE MINOR AXIS DIRECTION. AXIS DIRECTION. 40000 M THE MINOR AXIS DIRECTION. AXIS DIRECTION. 40000 M THE MINOR AXIS DIRECTION. AXIS DIRECTION. 40000 M THE MINOR AXIS DIRECTION. 4	U S	I R4	R 4	T##TRRVDOI	EC ALLE	APCL AK,	=	SOOF C4	((48(N
CTS) PSMAX AND SHIN ARE THE LOADS P(NFT=5 (AT/4) ************************************		2 N 9	8 1	IDDITTE SN	I LIII)		= : F = :	S	W I S A	((
S) SHAX AND SHIN ARE THE LOADS S(NFT=5 N FOR CIS *((AI/u)++f ₄ 5)+65537+((AI/N)++2 ₄ 5)=40 W)++4 ₄ 5) TE LOADING. DO AI DIRECTION? N2 DAR, ADK, AF, AT, AKHAX, AKMAXE, AKMAXO, AKM RPO, B, CCCP, CDAL, CCAR, CF, CI, CKMAX, CKM RPO, B, CCCP, CDAL, COAR, CF, CI, CKMAX, CKM AL, PHI, PI, OMA, COH, SHAX, SMIN, RAB, R AL, SMAX, YOUJ, SMIN(900) KMAX, AKMAXO, AKMIN, ALPHA, AMF, AO, AR, ARP CKMAXO, CKMAN, CN, CO, CC, CPT, CRPO, IA, T NC, NF, NI, NO, NP, PI SKC, SYS, M, WM, WN, Z COND KIND 2=AI+22)/CI++2)=(357643)*((CI+2=AI++ TION FFOR MINOR AXIS(AI) DIRECTION RT(PIAAI/OMAX) FFOR MAJOR AXIS CI) DIRECTION IF(AI+2/CI)/GMAX) TH(AI+2/CI)/GMAX) TH(AI+2/CI)/GMAX) TA(AI+2/CI)/GMAX) CONDING IN CI DIRECTION? OO HE MINOR AXIS DIRECTION? OO HE MINOR AXIS DIRECTION? CADING IN CI DIRECTION? CADING IN CI DIRECTION? CADING IN CI DIRECTION? CADING IN CI DIRECTION? CADING IN CI DIRECTION? CADING IN CI DIRECTION? COADING IN CI DIRECTION? CADING CI CI DIRECTION? CADING CAN CI CI CI CI CI CI CI CI CI						A A O O O O O O O O O O O O O O O O O O				Ţ
<pre>>PSMAX AND SHIN ARE THE LOADS=(NFT=5 FOR CIS (1A1/U)++T_15)+65557+((AI/W)++215)=10 >++415) LDADING: DIRECTION? AR,ADK,AF,AI,AKMAX,AKMAXE,AKMAXO,AKM PO.B.C.CCP.CDAL.CDAR,CF.CI.CKMAX.CKM F.H.PHI,FI,SMAX.OMIN,SHAX,SMIN,RAB,R F.H.PHI,FI,SMAX.OMIN,SHAX,SMIN,RAB,R F.M.PHI,FI,SMAX.OMIN,SHAX,SMIN,RAB,R CKMAXO,CKMIN,CN,CO,CK.CFFI,CRPO,IA,I CCNFT,NI,NO,NP,PI,SKC,SYS,M,WH,WN,Z OND KIND =AI++2)/CI++2)=(37/547)+((CI++2=AI++ ION EXYS)++2 SYS)++2 ALUATION FOR MINOR AXIS(AI) DIRECTION MANOR AXIS(CI) DIRECTION MANOR AXIS DIRECTION</pre>	ILEN I		I		C2 T//Y	DR ANK N	W	E D O A N	N * W	\$
PSMAX AND SHIN ARE THE LOADS P(NFT=5 OR CIS (1//W) ++f_5)+65577+((AI/W)++2_5)=10 ++4_5) LOADING_ DIRECTION? ADK, AF, AI, AKMAX, AKMAXE, AKMAXO, AKM O, B, CCCP, CDAL, CDAR, CF, CI, KMAXO, AKM A, B, CCCP, CDAL, CDAR, CF, CI, KMAXO, AKM A, PHI, PI, OMAX, OMIN, SHAX, SWIN, RAB, R SMAX(90U), SMIN(90D) AX, AKMAXO, AKMIN, ALPHA, AMF, AO, AR, ARP (MAXO, CKMIN, CO, CR, CRP I, CRPO, IA, I NFT, NI, NO, NP, PI, SKC, SYS, H, WM, W, Z, ND, KIND AI++2)/CI++2)=(37/647)+((CI++2=AI++ ON YS)++2 VAI++2)/CI++2)=(37/647)+((CI++2=AI++ ON YS)++2 LUATION_ OR MINOR AXIS(AI) DIRECTION (AI++2/CI)/OMAX) (AI++2/CI)/OMAX) (AI++2/CI)/OMAX) DIRECTION? MINOR AXIS DIRECTION MINOR AXIS DIRECTION? MINOR AXIS DIRECTION? MAJOR AXIS DIRECTION?	S	ê C E S	S	FTTF # # AUDE	OF ISSA	APC FL , MICC	=	001) ())))
MAX AND SHIN ARE THE LOADS [(NFT=5 R CIS I/W ++I_5)+65577+((AI/W)++2_5) [(0) ADDING RESS INTENSITY EVALUATION RESS INTENSITY EVALUATION ADK, AF, AI, AKMAX, AKMAXE, AKMAXO, AKM B,CICC+CDAL, CDA, CF, CICCHAXICKH , PHI, PY, JMAX, JMIN, SHAX, SWIN, RAB, R IAX (900), SMIN (900) AKMAXO, IAKMIN, ALPHA, AMF, AO, AR, ARP IAX (900), SMIN (900) AKMAXO, IAKMIN, ALPHA, AMF, AO, AR, ARP IAX (900), SMIN (900) AKMAXO, IAKMIN, ALPHA, AMF, AO, AR, ARP IAX (900), SMIN (900) AKMAXO, IAKMIN, ALPHA, AMF, AO, AR, ARP IAX (900), SMIN (900) AKMAXO, IAKMIN, ALPHA, AMF, AO, AR, ARP IAX (900), SMIN (900) AKMAXO, IAKMIN, CO, CR, CRP I, CRPO, IA, I FT, NI, NO, NP, PI, SXC, SYS, W, WH, WN, Z, AXO, CKMIN, CN, CO, CO, CRP I, CRPO, IA, I IX, IZ, CI, JMIN IX, IX, IND IX, IX, IND IX, IX, IX, IX, IX, IX, IX, IX, IX, IX,	== 1 E N	, C				R I I D I H S H X X X X	5 1		F0 ()	÷,
AX AND SHIN ARE THE LOADS P(NFT=5 /W)**T_5)+65577*((AI/W)**2_5)=40 &5) DINS_ RECTION?	I HS I		Ī	IIIN	*	A B F A F	R		RI4	M
X AND SHIN ARE THE LOADS (NFT=5 CIS J++T,5)+65577+((AI/W)++2,5)=40 SS INTENSITY EVALUATION ECTION? X,AF,AI,AKMAX,AKMAXE,AKMAXO,AKM CCCP(COAL,COAR,CF,CI,CKMAX,CKM HI,PT;OAAX,OMIN,SHAX,SMIN,RAB,R (904),SMIN(900) KMAXO,AKMIN,ALPHA,AMF,AO,AR,ARP O,CKMIN,CN,CO,CC,CFT,CCPO,IA,I NI,NO,NP,PI,SKC,SYS,W,WM,WN,Z, IND 2)/CI++2)=(37/647)+((CI++2#AI++ *2 *2 TON AI/OHAX AI/OHAX AI/OHAX SINC AXIS (AI) DIRECTION AI/OHAX AI/OHAX AI/OHAX SINC TION? IN CI DIRECTION? OR AXIS DIRECTION? OR AXIS DIRECTION? OR AXIS DIRECTION? IN CI DIRECTION? OR AXIS DIRECTION? IN CI DIRECTION? IN CI DIRECTION? IN CI DIRECTION? OR AXIS DIRECTION? OR AXIS DIRECTION? OR AXIS DIRECTION? OR AXIS DIRECTION?	ELI	J	R	H**H**G N	K + + + T		E	D RI	/1 _1	Ą
AND SHIN ARE THE LOADS (NFT=5 (S+T_5)+65577+((AI/W)++2_5)=40 (G_2 (IG_2 CTION? AF,AI,AKHAX,AKMAXE,AKMAXO,AKM (CCF,CDAL,CDAR,CFICI,CKMAXCKM (PT,OMAX,OMIN,SHAX,SMIN,RAB,R (OU),SMIN(900) AXCO,AKMIN,ALPHA,AMF,AO,AR,ARP (CKMIN,CN,CO,CR,CRPI,CRPO,IA,I (I,NO,NP,PI,SKC,SYS,W,WM,WN,Z, (O) (CT++2)=(37/547)+((CI++27AI++ (CT)/GHAX) (CT)/	= = = F T \ = =	0 F C F	EC	TNAA AA 0 F	1 2 **1		SS	I) E(51	x
ND SMIN ARE THE LOADS (NFT=5 +f_5)+6557+((AI/W)++2_5)=40 intensity Evaluation f, AI, AKMAX, AKMAXE, AKMAXO, AKM P; CDAL, CDAR, CF, CI CKMAX, CKM P; OAL, CDAR, CF, CI CKMAX, CKM P; OMAX, QMIN, SHAX, SHIN, RAB, R NU, SMIN(900) XO, AKMIN, ALPHA, AMF, AO, AR, ARP KMIN, CN, CO, CC, CRP I, CRPO, IA, I NO, NP, PI, SKC, SYS, W, WM, WN, Z, CT++2)=(37/647)+((CI++2#AI++ CI)/QMAX) CT++2)=(37/647)+((CI++2#AI++ AXIS (AI) DIRECTION CI)/QMAX) CI)/QMIN I AI DIRECTION? AXIS DIRECTION? AXIS DIRECTION? AXIS DIRECTION? AXIS DIRECTION? AXIS DIRECTION? AXIS DIRECTION?		2 2 T					5	N 5		A
D SHIN ARE THE LOADS P(NFT=5 1,5)+65577+((AI/N)++2,5)=40 ON? AI, AKMAX, AKMAXE, AKMAXO, AKM P,CDAL,CDAR,CF,CT,CT,CKMAXLCKM I,OMAX,QMIN,SHAX,SMIN,RAB,R),SMIN(900) O,AKMIN,ALPHA,AMF,AO,AR,ARP MIN,CO,CR,CRPT,CRPO,IA,I NO,NP,PI,SXC,SYS,W,HH,WN,Z, I++2)=(37/647)+((CI++2=AI++ AXIS(AI) DIRECTION MAX) AXIS(CI) DIRECTION I)/QMAX) I)/OHAX) I)/OHAX) I)/OHAX) AI DIRECTION XIS DIRECTION XIS DIRECTION AI DIRECTION STIC ZONE STIC ZONE AI DIRECTION	=	, ľ	I	ROARCC	, c	FCP	I	:	-	N
SHIN ARE THE LOADS (NFT=5 SHIN ARE THE LOADS (NFT=5 S)+65557+((AI/W)++2_5) TENSITY EVALUATION TENSITY EVALUATION AT AKHAX, AKMAXE, AKMAXO, AKM CDAL, CDAR, CF, CI CKMAX, CKM CDAL, CDAR, CF, CI CKMAX, CKM AX, CMIN, SHAX, SMIN, RAB, R SMIN(900) AKMIN, ALPHA, AMF, AO, AR, ARP IN, CN, CO, CR, CRP I, CRPO, IA, I O, NP, PI, SKC, SYS, W, WM, WN, Z, **2) = (37/647)*((CI**27AI** AXIS(AI) DIRECTION AXIS(CI) DIRECTION AXIS(CI) DIRECTION IS DIRECTION IS DIRECTION IS DIRECTION NT IS DIRECTION NT IS DIRECTION NT IS DIRECTION IS DIR	HSZI	J	0	HH ITA X	I	ANOVN		0	r	D
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 END END = SUBROUTINE FOR THE WHEELER RETARDATION MODEL = 1 SUBROUTINE WHEELR DOUBLE PRECISION ACP, ADAL, ADAR, ADK, AF, AI, AKMAX, AKMAXE, AKMAXO, AKMI EN, AKMINE, ALPHA, AO, AR, ARPI, ARPO, B, C, CCP, CDAL, CDAR, CF, Ct, CKMAX, CKMA EXO, CKMIN, CN, CO, CR, CRPI, CRPO, F, H, PHI, PI, OHAX, QMIN, SMAX, SMIN, RAB, PA EC, SKC, SYS, W, WH, WN, Z, BDAL, DDAL COMMON/TA1/A(B), JB(900), R(B), SMAX(900), SMIN(900) COMMON/TA2/ADAL, ADK, AF, AI, AKMAX, AKMAXO, AKMIN, ALPHA, AMF, AD, AR, ARPI C, CDAL, C, CDAL, CDK, CI, CKMAX, CKMAXO, AKMIN, ALPHA, AMF, AD, AR, ARPI E, IE, IP, J, JA, JE, KB, MN, MP, MR, NC, NFT, NI, NO, NP, PI, SKC, SYS, W, WM, WN, Z, A DDAR, CDAR, NT EVALUTION OF LINEAR CRACK GROWTH IN AIGCI DIRECTIONS EVALUATION OF RETARDED GROWTH IN THE AI DIRECTION ACP IS THE RETARDATION FACTOR IN THE AI DIRECTION WH IS THE WHEELER RETARDATION EXPONENT ACP=(ARPI/(AO+ARPO_AI))***** ADAR=ACP*ADAL IF(NFT_NE76) GO TO 900 EVALUATION OF RETARDED GROWTH IN THE CI DIRECTION, CCP IS THE RETARDATION FACTOR IN THE CI DIRECTION. CCP=(CRPI/(CO+CRPO+CI})**WH CDAR=CCP+CDAL RETURN END 900 ================= SUBROUTINE FOR THE WILLENBORG MODEL SUBROUTINE WILLNBG DOUBLE PRECISION ACP, ADAL, ADAR, ADK, AF, AI, AKMAX, AKMAXE, AKMAXO, AKMI EN, AKMINE, ALPHA, AO, AR, ARPI, ARPO, B, C, CCP, CDAL, CDAR, CF, CI, CKMAX, CKHA EXO, CKHIN, CN, CO, CR, CRPI, CRPO, F, H, PHI, PI, OHAX, OMIN, SHAX, SMIN, RAB, RA EC, SKC, SYS, NH, M, WN, Z, BDAL, DDAL COMMON/TAT/A(B), JB(900), R(B), SMAX(900), SMTN(900), SMTN(900) COMMON/TAT/A(B), JB(900), R(B), SMAX(900), SMTN(900), SMTN(900), SMTN(900), AR, ARPI CALL, CAL, NT SAR, CDAR, NT CALL GROWTH BDALSADAL SAR, SMAX,

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VI. SAMPLE CALCULATIONS

6.1. Introduction

Sample calculations using the program are done. The results are compared with the test results. Obviously, some discrepancy between the test results and the calculations are observed. Considering that the normal scatter in fatigue-crack-growth rates may range from a factor of two to four under identical loading conditions, these results are quite good.

6.2. Sample Calculations.

Broek (82) makes sample calculations with the stress history listed in TABLE 6.1. He uses the formula $K = 6\sqrt{\pi a}$ for the stress intensity factor evaluations. Obviously, this is applicable for center-cracked panels with infinite sizes, or when the crack length is very small compared to the width of the specimen. He uses Paris eq'n da/dN = 3.10^{-10} (ΔK)⁴ mm/cycle for linear (unretarded) crack growth evaluations. Despite the fact that the crack-growth rate corresponds to the crack length 2a for center-crack panel, some modifications are made in order to be able to compare the computer results with Broek's results.

TABLE 6.1.

Stress History

Number of cyles in the spectra (occurences)	S _{max} (kg/mm ²)	S _{min} (kg/mm ²)	Sys (kg/mm ²)
l	13.5	0	
9	12	0	
90	.9	0	50
900	5.8	0	
2000	2.8	0	

The results for the linear crack growth are approximately the same, but Broek's linear crack growth evaluation with fixed crack size per block is a slight overestimation.

Retarded crack growth results are very different. Broek assumes that crack length and crack growth rates are constant throughout the block cycles, to save computer time and so, expenditures. But this results in over-estimation of the fatigue life. Cycle by cycle integration results are much more accurate than block by block evaluation.

Why does Broek overestimate the fatigue life? Both the Wheeler and the Willenborg models show maximum retardation at the cycle which comes just after overload cycle. Retardation decreases as the crack length increases. Broek does not take into consideration that point. He assumes constant amount of retardation as long as the plastic zone of current cycle remains in the overload's plastic enclave. But, Broek makes that control just at the first cycle of the loading block. Although he considers retardation, there are some cycles which don't show retardation according to plastic enclave model. Of course this results in overestimation of fatigue life.

For constant amplitude loading; Simpson's composite integration scheme, the Wheeler and the Willenborg models are expected to show same estimations. Control was done with the program AKYÜREK-1. The Wheeler and the Willenborg models estimated the same results. Simpson's integration scheme with 14 steps estimates the result with an error 3.8 per cent. Doubling the number of steps decreases the discrepancy with the cycle by cycle integration scheme to 0.42 per cent error. Step size is chosen such small that error due to integration scheme is negligible.

Porter (83) makes tests with 2024-T3 center cracked panel (CCP) test specimen. Estimation using the Program AKYÜREK-1 for the stress spectrum, he designates as PlO, results in an overestimation with a factor of two. The main reason is the lack of accurate constant amplitude data for the specimen. In fact PlO is a good sample spectra for variable amplitude random sequence loading. On the other hand, since the overload ratios are so small that, changing the Wheeler exponent did not show much varience in the amount of retardation. There is another important point which should be carefully considered when using the program AKYÜREK-I;Stress or load spectra should be converted to stress or load history. But small errors in conversion may cause surprising results since the spectrum is repeated more than 2000 times for the case considered. For program PlO (83), the overestimated

cyclic life with a factor of two originates from uncertain knowledge of constant amplitude data. This was verified by program Pl (83). The program AKYÜREK-I overestimated the cyclic life with a factor of two. But, program Pl has constant K_{max} stress spectra. This should result in linear crack growth. It was proved that the program AKYÜREK-I predicted unretarded crack growth without error. That implies uncertain knowledge of constant amplitude data.

Fatigue crack growth test reports usually do not give the all required data about the test, such as yield strength, fracture toughness, specimen orientation, etc. Small variance in those data causes unexpected results. For example, because of the strain hardening process it is very difficult to obtain the yield strength of the specimen if it is not given. But, the retardation models operate according to plastic zone size which is strongly dependent upon yield strength.

On the other hand, in literature fatigue crack growth data are available for certain type of specimen and flaw, such as cracks originating from holes. It is necessary to write stress intensity factor equation for that special flaw type in order to be able to estimate cyclic life of the specimen and to compare the estimation with the test results. That can be done. But, it is not practical only to make such a comparison.

6.4. Restrictions and the Deficiencies of the Program

Of Course this program is in the stage of development, therefore it has some deficiencies and does not cover all types of loa dings and flaws. In fact, these deficiencies originate from the retardation models. Weakness of the retardation models is also weak ness of the program.

The Wheeler and the Willenborg models do not account for physical observations such as : (1) accelerated crack growth under certain load variations , (2) delayed retardation after a high peak load and (3) a sustained delay after a high peak load even when the crack has already fully penetrated through the plastic zone created by the high peak load.

Although compressive peak load causes accelaration of crack growth rate, the Wheeler and the Willenborg models do not account for this. Compressive stress intensities are set to zero. Johnson, W.S (84) modified the Forman's equation to consider the compressive loading :

$$\frac{C \Delta K}{(1-R^{eff})^{m} K_{c} - \Delta K}$$
(6.1)

where m = 1 at $R \ge 0$ and

m = 2 at $R \lt 0$

The equation is exactly that suggested by Forman except for the mexponent applied to the stress ratio. The exponent equals 2.0 for eff negative R values.

Robin, C. and Pelloux, R.M (85) made experiments to observe the crack growth retardation in an aluminum alloy. Figure 6.f shows that the test result of constant amplitude loading after a single overload. Figure 6.2 presents the crack growth rates calculated from the Wheeler and Willenborg models.



Distance from overload front, mm.

Figure 6.4. Plats of the crack growth rates against the crack length increment for $K_{max} = 16.5 \text{ MP}_{a} \text{ M}^{1/2}$ (K_L = 21.45 MP_a M^{1/2} O, center measurements :+, edge measurements.(83)



Distance from overload front, mm.

Figure 6.2. The crack growth rates calculated from the Wheeler and Willenborg models at $K_{max} = 16.5 \text{ MP}_a \text{ M}^{1/2}$

 $(K_{ol} = 21.45 \text{ MP}_{a} \text{ M}^{1/2})$: ____, o, Wellenborg model center measurements : ____, o, Wellenborg model, edge measurements ; ____, o, Wheeler model, center measurements ; ____, x, Wheeler model edge measurements.(85).

Considering Figures 6.6 and 6.7 together, the following observations are made.

(1) The two models do not predict the delayed retardation.

(2) The two models predict the same minimum crack growth rates for plane stress and for plane strain.

(3) The crack length increments over which retardation is predicted are smaller than the crack length increments over which retardation is observed. The two models give a better fit to plane stress data than to the plain strain data.

(4) The two models are quite similar and give a fair representation of the crack retardation phenomenon.

(5) Crack retardation is observed beyond the point at which the plastic zones are tangential. The microscopic crack growth da ta show that retardation exists until the crack front has reached the elastic-plastic interface of the calculated overload plastic zone.

(6) The two models are conservative, i.e., they predict less retardation than is observed.

The two models do not show the short period of crack acceleration after the overload that has been observed after large overload excursions. And, also they don't account for the difference in the retardation between a single peak overload and multiple peak overload

VII. CONCLUSIONS

In this thesis, a computer program evaluating the cyclic life of the specimens subjected to randomized block loading has been prepared. It uses the Wheeler and the Willenborg retardation models.

Either a cycle-by-cyle or block-by-block crack growth integration scheme is used for the fatigue life predictions.

In the program, center-crack panel, single and double edge cracked specimens, ASTM standard compact tension and bend specimens, and surface flawed plates are considered. Other type of flaws can be added to the program by making small changes.

Paris eq., Walker's formula and Forman's eq. are used for linear crack growth evaluations.

Three types of output options are available in the program. Print of the crack growth information can be obtained at the end of each loading block, spectra or every NP cycles (which is given as data). Simpson's composite integration formula is used for constant amplitude load fatigue evaluations. This requires least computer time among the predictive models. And, also good results can be obtained.

In the computer program AKYÜREK-I; no modification has been mede to the retardation models. Therefore, deficiencies of those models may result in overestimation or under_estimation of the fatigue life.

For a constant stress intensity range, crack growth rate increases with increasing cycle ratio $(=K_{\min}/K_{\max})$. Paris and Walker's crack growth equations don't consider that phenomenon, but only Forman's does. The crucial point of the random load fatigue life estimations is to predict the cyclic life from constant amplitude data. If the constant amplitude data are obtained at different cycle ratios and the program AKYÜREK-I is modified to take into account the different cycle ratios, more precise and better predictions can be made.

The most common fatigue flaw type is the surface flaw. Therefore, the main study has been devoted to surface flaws. Detailed information about that flaw type is given for the researchers desiring to study upon that subject.

Block by block crack growth integration scheme overestimates cyclic life of specimens. Since the predictions are sensitive to the Wheeler shaping (retardation) exponent especially in blockby-block integration scheme, better correlations can be done changing the shaping exponent.

A cycle by cycle crack growth integration scheme is favorable but it requires much computer time. To save computer time, block integration scheme can be chosen. A Range-Kutta integration routine should be used to determine the incremental crack growth for each series of discrete load levels. Thus, overestimation in cyclic life predictions can be avaided and the computer time and so, money can be saved. The program AKYÜREK-I can be modified to use a Range-Kutta integration routine, but that will require a tedious study.

Among the retardation models, only the crack closure model has some sort of physical basis. It can be put in the program AKYÜREK-I, as an alternative to the Wheeler and the Willenborg retardation models.

The Root-Mean-Square (RMS) retardation model is the simplest one and requires least computer time. It can be used in the program AKYÜREK-I, by using a new subroutine (e.g., SUBROUTINE RMS) and modifying the SUBROUTINE SIMPSN.

APPENDIX-A

THE METHOD OF LEAST SQUARES (MLS)

Let q_1 ,, q_n be n direct measured values of a magnitude, and let Q represent the sum of squares of their deviations from a number q :

$$Q = \sum_{i=1}^{n} (q_{i} - q^{2})$$
 (A.1)
i-1

The value of q making Q minimum, is the best value of the measurement, since Q involves squares of devlations which are non-negative. This best value of \bar{q} of q_1 ,..., q_n is shown to be the arithmetic of the measured values :

$$\frac{dQ}{dq} = -2 \sum_{i=1}^{n} (q_i - q) = \sum_{i=1}^{n} q_i - nq = 0$$

resulting.

$$\overline{q} = \frac{q_1 + q_2 + \cdots + q_n}{n}$$

(A.2)

(A.3)

 $\frac{d^2 Q}{d q^2} = 2n > 0$. Q is minimum for \overline{q} . This process of finding

 \bar{q} from eq(A.1) is known as the method of least squares(MLS).

Finding the best fitting involves the following steps : (i) From the distribution guess the type of the function as linear, quadratic, exponential,....

(ii) Write the general form of the function,

(iii) By the use of the MLS, determine the unknown parameters.

The distribution is as given in figure A.l. For instance, for the aspect ratio ; a/2C = 0.05

(i) We observe that the points $P_i(x_i, y_i)$ lie nearly on the function of the form $y = AX^r$.

(ii) Taking the common logarithm of both sides of $y = AX^{r}$, we get the linear equation.



Crack dept/specimen thickness ratio

Χ.

 $\log y = \log A + r \log x$ in logy and logx. Then setting u= logx, y = logy, s = logA we have :

(iii) By the MLS, we minimize

$$Q(s_{i}r) = \sum_{i=1}^{n} (v_{i} - ru_{i} - s)^{2}$$

in which s,r are the parameters.

$$\frac{\partial Q}{\partial r} = -2\sum_{i=1}^{n} (v_i - ru_i - s)u_i = 0$$

$$\frac{\partial Q}{\partial s} = -2 \sum_{i=1}^{\infty} (v_i - ru_i - s) = 0$$

which yields :

r

$$(\Sigma u_{1})r + ns = \Xi V_{1}$$

 $(\Sigma u_{1})r (\Sigma u_{1})s = \Sigma u_{1}V_{1}$

Solutions to these equations :

$$= \begin{vmatrix} \Sigma \mathbf{v}_{i} & \mathbf{n} \\ \mathbf{v}_{i} \mathbf{v}_{i} \mathbf{\Sigma} \mathbf{u}_{i} \end{vmatrix} / \Delta, \ \mathbf{s} = \begin{vmatrix} \Sigma \mathbf{u}_{i} & \Sigma \mathbf{v}_{i} \\ \mathbf{v}_{i} \mathbf{v}_{i} \mathbf{\Sigma} \mathbf{u}_{i} \end{vmatrix} / \Delta$$

where $\Delta = \begin{bmatrix} \Sigma u_{1} & n \\ \Xi u_{1}^{2} & \Sigma u_{1} \end{bmatrix}$

(A.4)

(A.5)

(A.6)

(A.7)

For the case considered, substitute $y = M_{k}$, x = a/B

then
$$M_k = A (a/B)^r$$
 (A.S)

As can be seen in figure A.1, there are eight curves. For the aspect ratios (a/2c) remaining between the curves, a linear in terpolation is used (see figure A.2).

$$M_{k} = M_{k1} + (M_{k2} - M_{k1}) \left\{ \left(\frac{a}{2c} \right) - 0.05 \right\}$$
(A.9)

where M_k is the value to be evaluated, corresponding to a/2c. M_{k1} and M_{k2} are the values corresponding to a/2c = 0.05 and a/2c = 0.10, respectively for the sample case.

Magnification factor, M_k , depends upon the angle ϕ . Since it is necessary to evaluate the stress intensity factors in the major axis and minor axis direction, the magnification factors in these directions are required. Smith (86) showed that the magnification factor in the direction of major axis of the elliptical flaw could be taken as 1. But this is an approximation (87).



Figure A.2. Linear interpolation for the aspect ratios remaining between the curves.



$$\begin{array}{l}
\mathbf{a} + 2hn \\
\int f(x)dx = \frac{h}{3} \left[f(a) + 4f(a+h) + 2f(a+2h) + 4f(a+3h) + \cdots \\
\mathbf{a} & 3 \\
+ f(a+2nh) \right] \\
\end{array}$$
(B.2)

Considering the exact matching method (figure B.2)



Figure B.2. Figure for the exact matching.

$$\int_{-h}^{h} f(x) dx = W_{1} f(-h) + W_{2}f(0) + W_{3}f(h)$$

for $f(x) \equiv 1$

$$\int_{-h}^{h} 1 \cdot dx = W_1 \cdot 1 + W_2 \cdot 1 + W_3 \cdot 1 \implies W_1 + W_2 + W_3 = 2h \quad (B.4)$$

for $f(x) \equiv x$

$$\int_{-h}^{h} x dx = W_{1}(-h) + W_{2} \cdot 0 + W_{3}(h) \Rightarrow -W_{1}h + W_{3}h = 0 \Rightarrow W_{1} = W_{3} \quad (B_{0} \cdot 5)$$

for $f(x) \equiv x$

$$\int_{h}^{h} x^{2} dx = W_{1}(-h)^{2} + W_{2} \cdot 0^{2} + W_{3}(h)^{2} \Rightarrow W_{1}h^{2} + W_{3}h^{2} = \frac{2h^{3}}{3} \quad (B.6)$$

from eqs $(B_{.4}), (B_{.5})$ and $(B_{.6})$

(B.3)

$$W_1 = W_3 = \frac{h}{3}$$
 and $W_2 = \frac{4h}{3}$

Substituting these values into $eq(B_{.3})$ yields :

$$\int_{-h}^{h} f(x) dx = \frac{h}{3} (f(-h) + 4f(0) + f(h))$$

or

1

$$b = d + 2h$$

 $\int f(x)dx = \frac{h}{3} (f(a) - 4f(a+h) - f(a+2h))$

for $f(x) \equiv x^3$

$$\int_{-h}^{h} x^{3} dx = W_{1} (-h)^{3} + 0 + W_{3} (h)^{3} \Rightarrow W_{1} = W_{3}$$

Simpson's formula is also exact for cubics.

Truncation error (E):

$$E = -\frac{h^2}{90} f(\theta) - Simple \text{ formula} \qquad (B.8)$$

$$E = -\frac{(b-a)}{90} h^4 f^{(1v)}(\theta) - composite formula \qquad (B.9)$$

(B.7)

APPENDIX - C

PROBABILITY - DENSITY DISTRIBUTION

Many engineering structures such as bridges, ships, and others are subjected to variable - amplitude random - sequence load fluctuations. The probability of occurence of the same sequence of stress fluctuations for a given detail in such structures obtained during a given time interval is very small. Consequently, the magnitude of stress fluctuations must be characterized to study the fatigue behavior of components subjected to variable - amplitude random - sequence stress fluctuations. The magnitude of the stress fluctuations should be characterized and described by analytic functions. The use of probability - density curves to characterize variable - amplitude cyclic - stress fluctuations appear to be very useful (69) .

Stress history, or stress spectrum, for a particular location in a structure subjected to variable - amplitude stress fluctuation can be defined in terms of the frequency of occurence of maximum (peak) stresses. Usually, frequency - of - occurence data are presentéd as a histogram, or bar graph (figure C.1), in which the height of the bar represents the percéntage of recorded maximum stresses that fall with in a certain stress interval represented by the width of the bar. For example, 20.2 % of the maximum stress in



Figure.C.l. Frequency-of-occurence data
figure. C.l.(a) fall within the interval between 7.5 and 8.5 ksi. The frequency of occurence of stress ranges can be represented by similar plots with the vertical scale changed according to the relationship between G_{\max} , G_{\min} , and stress range, G_{r} or ΔG . Since stress range is the most important stress parameter controlling the fatigue life of structural components, stress range is used to define the major stress cycles in the following discussion.

The frequency-of-occurence data can be presented in a more general form by dividing the percentage of occurence for each in terval, i.e., the height of each bar, in figure C.l.(a) by the interval width to obtain a probability-density curve such as shown in figure C.l(b). Thus, data from sources that use different st ress-range intervals can be compared by using the probability-density curve. The area under the curve between any two values of ΔG represents the percentage of occurence within that interval.

A single nondimensional mathematical expression can be used to define the probability-density curves for different sets of data. For example, Klippstein and Schilling (88) showed that the following nondimensional mathematical expression, which defines a family of skewed probability-density curves referred to as Rayleigh curves or distribution functions, can be used to accurately fit a probability-density curve to each available set of field data for bridges :

 $p' = 1.011 \times e^{-\frac{1}{2}(x')^2}$

95

(c.1)



Figure C.2. Characteristics of Rayleigh probability curves where x' = (Gr - Grmin)/Grd, Gr(=AG) is the stress range and Grmin (i.e. $\Delta Grmin$) and Grd (i.e. AGd) are constant parameters that define any particular probability-density curve from the family of curves represented by eq (C.1). Eq (C.1) is plotted in figure C.2(a). As illustrated in figure C.2(b), a particular curve from the family is defined by two parameters : (1) the modal stress range , Grm, which corresponds to the peak of the curve ; and (2) the parameter Grd, which is a measure of the width of the curve or the dispersion of the data. The curve could be shifted sideways by changing Grm, and the width of the curve could be modified by changing Grd. Mathematical expressions for the modal, median, mean, and root-mean-square values of the spectrum are given in figure C.2. The rootomean-square (rms) value is defined as the square root of the mean of the squares of the individual values of x^2 or Gr (70).

APPENDIX-D

AKYUREK - I USERS MANUAL

D.1. Introduction

The computer program AKYUREK - I predicts the fatigue life of the specimens subjected to randomized block loading . It uses the Wheeler and the Willenborg retardation models. Center cracked, edge cracked, surface flawed specimens, standard compact tension and bend specimens are considered . Paris - Erdoğan, Forman and Walker's linear crack growth calculation formulae are used. Either cycle - by - cycle or block - by - block crack growth integration scheme can be chosen.

AKYÜREK - I is in its developmental stage in terms of new options, more sophisticated models, etc. The program, however, has been designed to be modular in organization which permits additions and changes to be made more easily.

D.2. The Program Outline

The program consists of fifteen routines and a supervisory routine. The outline and simplified flowchart is given in Chapter V (pp .60 - 63)

D.3. Data Preparation

<u>CARD SET]:</u> Selection of the retardation model (I 10) - One card. Columns: (1 - 10) MR Model of retardation.

- (=1 Wheeler model)
- (= 2 Willenborg model)
- (= 3 Const. amplitude Loading-Simps

integration scheme)

Columns : (11 - 20) MI Model of integration scheme . 1. (=1 Cycle by cycle crak growth. integration scheme) NOTE : if MR = 3 then MI=2 (= 2 Block by block integration scheme CARD SET 2: Title card (20 A 4) - one card. Columns : (1 - 72) TITLE Title of the problem. It is limited to 72 alphanumeric characters. CARD SET 3: Bounds on the problem (2020. 14) - one card. Columns : (1 - 20) AI Intial crack length. Final (or critical) crack length . (21-40) AF (If the crack length growess to final crack length, the program stops.)

<u>CARD SET 4</u>: Maximum and Minimum stresses (2D20.14) - one card. If MR is not equal to 3, Omit this card.

Columns : (1 - 20) SMAX Maximum stress. (21 - 40) SMIN Minimum stress.

If NFT = 4 or 5 SMAX and SMIN are loads (see card set \mathscr{G}_{-})

CARD SET 5 : Selection of the Linear crack growth eq'n (I10) - one card. Columns : (1 - 10)MN Linear crack growth equation number (=1)Paris - Erdoğan equation) (=2 Forman's equation) (=3 Walker's calculation formula) CARD SET 6: Linear crack growth data and material properties (3D20.14) - two cards. Card 1 Col's : (1 - 20)Material const. in Paris eq., or С Forman's eq. or Walker's formula. Such as in $C(\Delta K)^{CN}$. (21 - 40)CN Material const. in Paris/eq., or Forman's eq. or Walker's formula . Such as in $C(\Delta K)$. (41 - 60) SKC Fracture toughness of the material. col's : (1 - 20)Yield strength of the specimen . 2 SYS card (21 - 40)W Width of the specimen. (41 - 60)Thickness of the specimen . В (61 - 70)NFT Number of flaw type. (=1 Center - cracked panel) (=2 Single edge notched specimen) Double edge cracked specimen) (=3)

> (=4 Standard bend specimen) (=5 compact tension specimen)

(=6 Surface flawed specimen)

<u>CARD SET 7</u>: Material property (D20.14) - one card If MN is not equal to 3 omit this card (see card set 5)

Columns: (1 - 20) WN Walker number. (Exponent in the Walker's formula. da/dN = $CK_{max}^{WN} \Delta K^{CN}$)

<u>CARD SET 8</u>: Flaw size (D20.14) - one card. If NFT is not equal to 6 omit this card. (see card set 6) Columns : (1-20) CI half the crack length of the surface

flaw in the major axis direction.

CARD SET 9: Specimen size (D20.14) - one card. If NFT is not equal to 4 omit this card. (see card set 6) Columns : (1-20) Z Distance between the supports in the

bend specimen.

NOTE: If MR is equal to 3 omit all cards below I

<u>CARD SET 10</u>: Control data (D20.14, 3I10) - one card . Columns : (1 - 20) ALPHA the constant required to evaluate the plastic zone size .

102 .

	1 - A - A	
		(= 3. plane strain case)
 · · · · · ·		(= 1. , plane stress case)
(21 - 30)	MP	Number of the plastic zone formulation.
		(=1 Dugdale model)
		(= 2 Irwin model)
(31 - 40)	JE	Total number of flights, spectra or
		repeating groups of loading blocks.
(41 - 50)	JA	Total number of blocks.
	•	

<u>CARD SET 11</u>: Control of the retardation (D20.14) - one card. If MR is not equal to 1 omit this card.

Columns : (1 - 20) WM Wheeler exponent (retardation exponent) WM is chosen such that, similar results to the test data could be obtained .

<u>CARD SET 12</u>: Stress or Load history input -(I10,2D20.14)- JA cards.

Columns	: (1 -10)	JB(IA)	Total number of cycles in the IA [*] th block
•	(11-30)	SMAX(IA)	Max. stress in the IA 'th block
	an An an an an an an an an an an an an an an		If NFT=4 or 5 -> Max. load
	(31-50)	SMIN(IA)	Min. stress in the IA' th block
	• •		If NFT=4 or 5 - min. load

<u>NOTE</u>: If number of cycles in a flight or total number of blocks exceeds 900, Dimensions of JB, SMAX and SMIN in the COMMON blocks should be increased. Without any change in dimensions, JA can at most be 900.

CARD SET 13: Output control data (I10) - one card.

columns: (1 - 10) IP

Printing type

(=l prints out information at the`
end of each block)

(= 2 prints out information at the end of each flight, spectra or group blocks)

(=3 prints out information at the end of every NP cycles. See card set 14).

<u>NOTE</u>: If block by block integration scheme is used, output is in a fixed form. But, You Should write output control card only to satisfy data correspondance.

<u>CARD SET 14</u>: Print control (I10) - one card. If IP is not equal to 3 omit this card.

columns : (1 - 10) NP Information is printed at the end of every NP cycles.

NOTE : Read the note written above.

D.4. Dictionary

Magnification factor curve fitting parameter.

AMF = A * (RAB) * R, RAB is crack length-thickness ratio in surface flaw ACP Retardation factor in the AI direction ADAL Linear crack growth rate in the AI direction Retarded crack growth rate in the AI direction. ADAR ADK Stress intensity range in AI direction Final or critical crack length in AI direction AF AI Initial or current crack length. AKMAX Maximum stress intensity factor in AI direction. AKMAXE Maximum effective stress intensity factor in AI direction. AKMAXO Maximum stress intensity factor due to overload. AKMIN Minimum stress intensity factor in AI direction. AKMINE Minimum effective stress intensity factor in AI direction. constant. =1 for plane stress and =3. for plane strain. ALPHA Α Magnification factor in AI direction in surface flaw. AMF AO Crack length at the overload applied. AR Cycle ratio in AI direction AR = AKMIN/AKMAX ARPI Current plastic zone size in AI direction. ARPO Plastic zone size due to overload in AI direction. В Thickness of the specimen. С· Material constant used in linear crack growth equations. Wheeler retardation factor in CI direction. CCP Linear crack growth rate in CI direction. CDAL

CDAR	Retarded crack growth rate in CI direction.
СФК	Stress intensity range in CI direction.
CF	Stress intensity correction factor polynomial.
CI	Current or initial crack length in the major axis direction
	in surface flaw.
ĊKMAX	Max stress intensity factor in CI direction.
CKMAXE	Max. effective stress intrensity factor in CI direction.
СКМАХО	Max. stress intensity due to overload in CI direction.
CKMIN	Minimum stress intensity factor in CI direction.
CKMINE	Minimum effective stress intensity factor in CI direction.
CN	Material constant used as exponential in linear crack growth
	equations.
CO	Crack length at overload, in CI direction.
CR	Cycle ratio in CI direction . CR = CKMIN/CKMAX.
CRPI	Current plastic zone size in CI direction.
CRPO	Plastic zone size due to overload, in CI direction.
F	Function used in Simpson's formulation's.
H	Step size used in Simpson's formula.
IA	Current block number. IA=1, JA.
IB	Current cycle number in the block. IB=1, JB(JH).
IE	Current flight or spectra or group of blocks number.
	IE=1, JE .
IP .	Print control data . IP = 1 prints out at the end of each
	block, IP=2 each flight, IP=3 every. NP cycles.
JA	Total number of blocks in a group or total number of cycles
•	in a flight.

JE	Total number of flights, spectra or repeating groups of
· ·	blocks.
JB(JH)	Total number of cycles in JH - th block
MI	Model of integration scheme.
	MI=1 cycle - by - cycle int.
	=2 block - by - block int.
MN	Linear crack growth equation number .
	MN=1 Paris - Erdoğan equation.
	=2 Forman's equatión.
	= 3 Walker's calculation formula.
MP	Plastic zone size evaluation model number.
•	MP=1 Dugdale model, =2 Irwin model.
MR	Retardation model number . MR=1 Wheeler model
	= 2 Willenborg model
·* .	=3 Simpson's formula.
NFT	Number of flaw type.
	NFT = 1 center crack panel.
· · · ·	= 2 Single edge crack
	= 3 Double edge crack
· · · · ·	= 3 Double edge crack= 4 Bend specimen
· ·	 = 3 Double edge crack = 4 Bend specimen = 5 Compact tension specimen
	 = 3 Double edge crack = 4 Bend specimen = 5 Compact tension specimen = 6 Surface flaw.
NI	 = 3 Double edge crack = 4 Bend specimen = 5 Compact tension specimen = 6 Surface flaw. Data set number for input. NI = 5
NI NO	 = 3 Double edge crack = 4 Bend specimen = 5 Compact tension specimen = 6 Surface flaw. Data set number for input. NI = 5 Data set number for output. NO = 6
NI NO NP	 = 3 Double edge crack = 4 Bend specimen = 5 Compact tension specimen = 6 Surface flaw. Data set number for input. NI = 5 Data set number for output. NO = 6 Print is made at the end of every NP cycles.
NI NO NP NT	 = 3 Double edge crack = 4 Bend specimen = 5 Compact tension specimen = 6 Surface flaw. Data set number for input. NI = 5 Data set number for output. NO = 6 Print is made at the end of every NP cycles. Total number of cycles.

PHI	Elliptic integral of the second Kind.
Ρ́Ι	PI = 3.1415927
QMAX	Flaw shape parameter due to maximum stress.
QMIN	Flaw shape parameter due to minimum stress .
R	Constant used as exponent in magnification factor evalution
	in surface flaw.
RAB	Crack length - thickness ratio. RAB = AI/B.
RAC	RAC = AI/2 * CI.
SKC	Fracture toughness of the specimen.
SMAX	Maximum stress.
SMIN	Minimum stress.
SYS	Yield strength of the specimen.
W	Width of the specimen.
WM	Wheeler retardation exponent.
WN .	Walker's number used as exponent in Walker's calculation
	formula.
Z	Distance between the supports in bend specimen.
•	

D.5 Sample data preparations

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D.6. Restrictions and Limitations.

Certain restrictions have been imposed upon the analysis.

a. Compressive loads

The theory behind the crack growth rate laws and the retardation models was essentially developed for tension - tension loading . Consequently all compressive loads in the spectrum are trancated to zero.

b. Surface flaws

Surface flaws are grown under the assumption of constant shape. The current AKYÜREK - I surface flaw formulation and the magnifica tion factor are only valid for $a/2C \leq 0.5$.

c. Unit system.

There is no restriction for unit system. But the data used in the crack growth evaluations should be consisted in. Output is set to the units of data . For example , if a , is in inches, the output of (da/dN) lin, (da/dN) ret. ai, ao are all in inches.

d. Block by block crack growth integration scheme. This option is available only to have approximate information about fatigue behaviour of the specimens subjected to randomized block loading . Since linear or retarded crack growth rate is assumed to be constant within the block , block by block integration scheme overestimates the cyclic life . The smaller the block size, the better the results it estimates . This integration scheme is not used for surface flaws.

APPENDIX - E

SPECIMEN ORIENTATION NOTATION.

Some of the test specimens are machined from large plates. The specimens from the same plate show different crack propagation rates under identical conditons, if their orientations are different. The orientation of a specimen is expressed with two_letters (figure_E.1) The first letter is the direction of the normal of crack plane. The second letter stands for the crack propagation direction.

- L. Longitudinal
- T. Transverse (long)
- S. Short transverse

Figure E.1. Specimen orientation notation for plate materials.

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